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DESIGN OF A 4½ STAGE TURBINE WITH A STAGE LOADING FACTOR OF 4.66 AND HIGH SPECIFIC WORK OUTPUT

P. F. Webster

Prepared by
PRATT & WHITNEY AIRCRAFT
East Hartford, Conn. 06108
for Lewis Research Center



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SUMMARY

A highly loaded, high work four-stage turbine has been aerodynamically designed with an axial inlet and with exit guide vanes to eliminate exit swirl with minimal loss. This design features controlled vortex flow, terraced inner flowpath, transitional boundary layer flow on each airfoil row, localized airfoil recambering, and zero seal leakage flow. The result is a turbine with high levels of gas turning and Mach numbers relative to a conventional design.

The predicted efficiency for this turbine is 89.3% without the exit guide vane.

INTRODUCTION

The application of highly loaded, high work fan-drive turbines is found in advanced subsonic cruise and lift fan engines. These engines must be lightweight, have high overall performance, and meet low noise requirements. In these types of engines high by-pass ratios are used to improve cycle performance, which results in large fan diameters and increased work requirements for the low pressure turbine. The combination of low rotative speeds and increased work output in the low pressure turbine necessitates the use of high stage loading and high specific work technology to keep engine weight, size and complexity at a minimum. A study conducted at Pratt & Whitney Aircraft of the engine for an advanced transport airplane indicated that a 4½ stage turbine (4 stages plus exit guide vanes) with a stage loading factor of 4.66 and high specific work output would be required to drive the fan. If conventional stage loading factors (2.0 or lower) were used, this turbine would consist of 8 or more stages.

The objective of this program is to design and fabricate the 4½ stage turbine and to determine its performance in a cold air investigation. This report describes the initial phase of the program, the turbine aerodynamic design.

TURBINE AERODYNAMIC DESIGN

The turbine aerodynamic design encompasses the discussion of the turbine requirements, the design philosophy and the turbine flowpath, velocity diagrams, and airfoil definitions.

DESIGN REQUIREMENTS

The aerodynamic design parameters required for this turbine design approximate those resulting from a Pratt & Whitney Aircraft engine study for an advanced transport airplane application. These requirements are summarized as follows:

Number of Stages	n	4 Plus Exit Guide Vanes
Average Stage Load Factor	$\frac{g J \Delta h}{nUm^2}$	4.66 Based on Root Mean Square Pitch Diameter

Equivalent Specific Work	$\frac{\Delta h}{\theta}$	104,430 Joules/kg (44.9 BTU/lb)
Equivalent Rotative Speed	$\frac{N}{\sqrt{\theta}}$	2980 RPM
Equivalent Mass Flow	$\frac{W\sqrt{\theta}}{\delta}$	6.078 kg/sec (13.4 lb/sec)
Equivalent Mean Diameter	$D_{\mathbf{m}}$	48.006 cm (18.90 in.)

The last three of the requirements result from applying a 0.5 linear scale factor to the study engine turbine to make the cold air turbine compatible with existing NASA Lewis test facilities. The turbine was designed to be investigated with inlet total state conditions of 422°K (300°F) and 1.565 atmospheres. These test conditions duplicate the study engine turbine Reynolds number at cruise flight conditions. The turbine was also specified to have an axial inlet flow so that the test results could be compared to those obtained from previous tests of turbines designed with high stage loading factors. No requirements were made on the turbine exit guide vanes (denoted EGV hereafter) other than the elimination of exit swirl.

DESIGN PHILOSOPHY

The turbine design personnel at Pratt & Whitney Aircraft believe that the achievement of an efficient turbine design for this application requires the incorporation of certain aerodynamic concepts into the turbine design. These concepts are discussed in the following paragraphs.

Controlled Vortex Flow

The design of a highly loaded, high specific work low pressure turbine presents the problem of obtaining adequate levels of root reaction especially when a free vortex flow design philosophy is used. Reaction is defined as the ratio of the static pressure change across the moving blade row to the static pressure change across the turbine stage. It has been found that low stage reactions lead to extreme pressure gradients on the suction surface of the airfoil which increases the danger of boundary layer separation. For this turbine, a controlled vortex flow principle was used which enhances the design by altering the spanwise work distribution to increase the root reaction and decrease the tip reaction. The reduction in tip reaction also is beneficial since this reduces the potential for blade tip leakage. The use of controlled vortex flow as a design tool is discussed in Reference 1, and has been incorporated into the turbine design procedure used at Pratt & Whitney Aircraft.

Terraced Inner Diameter Flowpath

In a typical controlled vortex design, the vane exit angle is redistributed, relative to free vortex flow by increasing the vane root angle and decreasing the vane tip angle. This results in a reduction in the vane exit static pressure gradient. Investigation into the effects of flow-

path inner wall geometry on streamline curvature has shown that the combination of a conical vane endwall followed by a cylindrical blade endwall (referred to herein as "terraced") forces the local streamline curvatures in a way which also decrease the vane exit radial static pressure gradient. However, with a terraced flowpath, the reduced radial pressure gradient is accompanied by a reduction in streamline diffusion through the following blade row which does not occur with the vane angle redistribution. It is believed that the reduction in diffusion may result in reduced secondary flow losses. Therefore, this turbine design incorporates a combination of the vane exit redistribution and a terraced inner flowpath.

Transitional Boundary Layer

Experimental cascade research conducted by H. Schlichting and A. Das (2), H. Hebble (3) and K. Gersten (4) have clearly established that the profile performance of cascades in the Reynolds number range from 1.5×10^5 to 13.0×10^5 is strongly influenced by the nature of the transition from a laminar to a turbulent boundary layer on the airfoil suction side. As the Reynolds number decreases from a level where the suction side boundary layer is primarily turbulent, the laminar boundary layer transition moves further downstream towards the minimum pressure point. Upon reaching a Reynolds number where transition does not occur naturally before the minimum pressure, it appears that the boundary layer undergoes transition in a laminar separation bubble and reattaches to the profile surface as a turbulent boundary layer. At still lower Reynolds numbers, reattachment fails to occur, and the profile is separated in the classic sense.

A new calculation procedure has recently been developed by Pratt & Whitney Aircraft which accounts for the transitional nature of the airfoil boundary layer, the end product being a predicted airfoil profile loss. The results of this calculation are shown in Figure 1 for a typical airfoil design. A predicted turbulent boundary layer solution is also shown which has been used as a base for design performance estimation. As can be seen, the change in the boundary layer characteristics are reflected in the airfoil profile loss, i.e., the loss decreases as the transition point moves toward the minimum pressure point. The loss then increases gradually until boundary layer reattachment fails to occur. Therefore a minimum loss exists for each airfoil which is dependent on the transitional nature of the boundary layer, and which has been verified by cascade performance testing. The present four stage turbine design aims to capitalize on this loss characteristic by designing to an airfoil Reynolds number in the minimum loss region of each row through the variation of the airfoil chord.

Turbine Endwall Loss

The presence of divergent endwalls, high airfoil turning and high Mach numbers in a turbine design may result in large endwall losses. One of the mechanisms believed to cause end loss is the build up of the boundary layer as the fluid flows through the airfoil channel. Because of the pressure gradient across the channel, the lower momentum fluid is drawn to the interface of the suction surface and the inner wall where it forms a strong vortex, resulting in high losses. An investigation was made into the effects of airfoil recambering in the root endwall to reduce the cross channel pressure gradient. The results of this study are discussed in Reference 5, but the main conclusion was that a local recambering of the turbine airfoil by reducing the airfoil inlet and exit metal angle can lead to loss reductions in the root end-

wall regions. This approach has now been used in the 4½ stage turbine design between the root and quarter root sections.

Inner and Outer Seal Leakage

Gas path leakage past the rotating blade tip seals and the inner diameter spacer seals is known to have a large impact on turbine efficiency. One method for minimizing these leakage flows has been the use of abradable seal land materials that make very small running clearances possible. Since abradable seals are now being used in many turbine applications, the assumption of near zero leakage for the inner and outer seal configurations has been made.

DESIGN PROCEDURE

Meanline Analysis

The turbine flowpath and stage work distribution were set as a result of a parametric study using a Mean Line Design analysis approach. The Mean Line Design analysis offers a relatively fast meanline calculation by which a variety of turbine configurations can be compared and the optimum configuration selected. For this design, the inlet annulus was sized to give an inlet Mach number of 0.3 without inlet swirl. The fourth stage exit annulus was initially set to give 30° of swirl and a 0.46 exit Mach number. A parametric study was then carried out which varied the exit annulus area and stage work split while maintaining the average turbine mean diameter at 48.006 cm (18.9 inches). Varying the annulus area results in a trade-off between airfoil turning and turbine Mach number. The meanline results giving the effect of annulus area variation on efficiency are shown in Figure 2 for a stage work split of 27%, 27%, 27%, 19% in stages 1 through 4 respectively. The efficiency levels in this figure do not represent the final turbine design. This curve indicates that little improvement in efficiency is obtained by increasing the annulus area more than 15% above the base.

The variation of the fourth stage work determines the optimum overall turbine efficiency including the exit guide vane turning loss. The remainder of the turbine work was split equally in the first three stages. The exit guide vane loss was calculated using diffusion factors of 0.4 and 0.6, and the loss correlation from Reference 6. The meanline analysis results for this study are also shown in Figure 2 for the 15% increased exit annulus turbine. These curves indicate that the optimum turbine efficiency including the EGV loss would be obtained by designing the turbine with a near equal work split (25.5%, 25.9%, 24.5%, 24.1%) and a diffusion factor of 0.6. This means that the exit guide vane will be a highly loaded cascade with average inlet swirl of 45°. A reduction in the diffusion factor would result in the necessity to shift work out of the fourth stage, producing low fourth stage root reactions and high losses.

Streamline Analysis

Having determined the basic turbine flowpath, stage work split and the average (meanline) velocity triangles, the radial distribution of aerodynamic properties was determined using the Streamline Design Analysis. The basis of this analysis is a streamline design computer program which accounts for the radial component of gas velocity by using a streamline cur-

vature solution to the equation of motion for an axisymmetric, inviscid, compressible flow. The most important aspect of the streamline procedure is the ability to use controlled vortex principles to maximize the turbine efficiency.

The results of the Meanline Design Study provide predicted airfoil profile and endwall loss levels which are distributed across the span by using Pratt & Whitney Aircraft's design experience. The airfoil chords were set to obtain airfoil Reynolds numbers between 3.0 and 4.0 x 10⁵. This Reynolds number range was found to give a minimum profile loss in the cascade test mentioned earlier. The root and tip reaction levels were adjusted by the use of controlled vortex flow through a redistribution of vane exit angles and the "terracing" of the inner flowpath. A 30% to 40% root reaction level in all airfoil rows was the goal. Once a satisfactory solution was obtained, the turning in the airfoil roots was reduced to relieve the endloss as previously discussed.

The results of the streamline analysis are shown in the flowpath of Figure 3 and the velocity triangles at five radial locations in Figures 4 and 5. The flowpath is convergent through the exit guide vane to reduce the local diffusion factor. The number of turbine airfoils shown in Figure 3 was based on the application of the compressible form of the Zweifel lift coefficient in the range of 0.9 and 1.1 which is within Pratt & Whitney's experience for good profile efficiency. A tabulation of aerodynamic properties from the streamline analysis is given in Table I for the root, mean and tip sections, with typical airfoil nomenclature shown in Figure 6.

Airfoil Design

The airfoil contours were designed through an analysis of the airfoil surface pressure distribution, channel convergence, and surface boundary layer behavior. An airfoil design computer program generates suction and pressure surfaces based on input geometric parameters such as inlet and exit gas angles, solidity, throat dimensions and leading and trailing edge thickness. Inlet metal angles were determined by applying 3° to 6° of negative airfoil incidence, which is based on Pratt & Whitney Aircraft's experience on high performance turbine airfoils. Exit metal angles were determined by applying a gas angle deviation criterion derived from design experience, and influenced by the airfoil exit Mach number and gaging angle. Trailing edge diameter was to be 0.0253 cm (0.010 in.) to maintain the 0.5 scale factor between the cold air turbine and engine turbine. However, this diameter was increased to 0.038 cm (0.015 in.) to reduce fabrication costs.

Once the airfoil contours are defined, a surface pressure distribution is calculated by means of either a computer program which calculates the two-dimensional, irrotational flow of a perfect, compressible, inviscid gas through an entirely subsonic airfoil channel, or a program which applies a transient technique of the conservation laws to small control volumes in the flow field for a transonic airfoil channel.

The transient technique permits a mixed flow solution without advance knowledge of the interface regions of subsonic and supersonic flows. The resulting pressure distributions are then appraised on the basis of accelerating suction surface velocity while the suction surface

diffusion is minimized in the uncovered regions where the airfoil is susceptible to boundary layer separation. The channel geometry is also reviewed to ensure that a minimum amount of channel diffusion exists near the airfoil leading edge. This is to prevent the occurrence of a pressure side separation and reattachment with a subsequent increase in profile loss.

Surface boundary layer behavior of each airfoil defining section is also examined for possible separation through a program which uses a finite difference procedure to compute the laminar, transitional, and turbulent development of the airfoil surface boundary layer. This problem also calculates momentum and displacement thicknesses for the suction and pressure sides at the trailing edge which are used in a control volume wake mixing computation for determining the profile loss.

The results of the airfoil design procedure, i.e., the airfoil contours, pressure and velocity distributions and channel area ratios are presented in Figures 7 through 86. A summary of the defining section airfoil geometry is given in Table II, and the associated airfoil nondimensional coordinates in Table III. The results of the total pressure loss calculations for the mean sections over a range in Reynolds numbers are shown in Figure 87. This figure indicates that the selection of airfoil chord to give Reynolds numbers between 3 and 4.0 x 10⁵ does result in minimum losses in most airfoil rows.

EXIT GUIDE VANE DESIGN

The establishment of an equal work distribution in the turbine stages results in the need for a highly loaded, high turning exit guide vane design. The exit guide vane was therefore designed using fan exit guide vane technology based on stator data from the NASA high tip speed, low tip speed and 1800 fps tip speed fan programs discussed in References 7, 8 and 9 respectively.

The requirements for the exit guide vanes, as determined from the streamline analysis, are shown in Figure 88 which give the inlet angle and inlet Mach number distributions. The EGV airfoil sections were chosen to be 65 series thickness distributions in circular arc meanlines (656A) and were designed on conical surfaces approximating a stream surface of revolution. These sections were chosen to best accommodate the anticipated range of Mach numbers and Reynolds numbers. The resulting EGV design has 28 vanes an an 8.636 cm (3.4 in.) true chord which is constant spanwise. The aspect ratio is 1.6 based on average blade length and an axially projected chord at the hub. Maximum thickness to chord ratio varies linearly with radius from 0.06 at the hub to 0.10 at the tip. The convergence of the flowpath is the result of iterations aimed at controlling root loadings and wall diffusion rates while maintaining levels of solidity, Mach number and aspect ratio within P&WA experience. The incidence angles were set to correspond with past experience and to provide some additional choke margin for off-design requirements while the EGV deviation angles were determined by applying Carters Rule plus an adjustment based on P&WA experience. The EGV incidence and diffusion factors are shown in Figure 89 as a function of span and the resulting airfoil contours are shown in Figure 90 for 5 radial locations. Non-dimensional coordinates for the EGV are given in Table IV. EGV surface pressure distributions were also computed by means of a compressible potential flow solution program per T. Katsanis (Reference 10).

These results are shown in Figures 91 through 95, and indicate satisfactory flow at all sections.

The EGV losses were calculated by using a correlation of loss parameter versus diffusion factor and percent span. The correlation as reported in Reference 11 was modified slightly to represent more closely the stator data from other NASA fan programs. Because the Reynolds number at the design conditions is lower than those normally encountered in compressor designs, the loss calculation was adjusted assuming loss proportional to Re $^{-.2}$ and using Re = 10^6 for the parameter data. The resultant loss is shown in the spanwise curve of Figure 89, the average loss being 2.1% $\Delta P_T/P_T$.

EFFICIENCY ESTIMATION

The result of the Meanline Design and Streamline Studies was the establishment of the turbine flowpath, work distribution and velocity diagrams. The estimated efficiency for this turbine based on those diagrams is 88.1% for airfoils with 0.010" trailing edge thickness, exclusive of the EGV loss. Increasing the airfoil trailing edge diameter to 0.015" reduces this efficiency to 87.9%. Application of the minimum profile loss assumed for the transitional boundary layer design procedure will result in an improvement of 1.4% to 89.3%. The exit guide vane loss of 2.1% would then give an overall turbine efficiency with EGV of 88.6%.

Airfoil Vibration Analysis

The airfoils defined in this report have been analyzed for vibratory resonance and airfoil flutter. The vibratory analysis idealizes one blade as vibrating in a complete rotor stage. The turbine disk is modeled via thin disk and ring equations. The shroud is modeled with equations as a continuous ring. The root and disk dead rim flexibilities are entered as connector springs between disk and blade. The disk and shroud are then assumed to undergo a sinusoidal vibratory mode with an integral number of waves around the rim and the system natural frequency is then found.

The results of this analysis are shown in the resonance diagrams of Figures 96 through 99 for each rotor. The first stage blade is shown to be free of all critical resonances within the anticipated steady-state operating range of the turbine. (±25% of design speed). Rotors 2 through 4 do have nozzle passing frequency resonances within the operating range. These resonances are first and second bending, and first torsional modes of vibration. The magnitude of the vibratory stress levels are not known due to the lack of engine experience with airfoils of such high thickness to chord ratios, and high camber which make for a stiff airfoil. However, engine experience on conventional LPT airfoil designs has resulted in vane passing vibratory stress levels of 7720 kg/m² (11.0 ksi) which is half of the vibratory stress capability as shown on the Goodman Diagram of Figure 100. Mechanical damping and the low pressure levels of a cold flow rig versus an engine will also minimize the excitation energy. Therefore no vibratory resonance problems are anticipated for these airfoils.

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SYMBOLS AND DEFINITIONS

A area $(cm^2, in.^2)$

B axial chord (cm, in.)

C_I compressible lift coefficient

C_x axial velocity (m/sec, ft/sec)

D diameter (cm, in.)

D_f diffusion factor for incompressible 2 dimensional cascade

ΔH turbine total work (joules, BTU)

Δh turbine specific work (joules/kg, BTU/lbm)

L blade or vane height at airfoil throat (cm, in.)

LED leading edge diameter (cm, in.)

M Mach number

n number of stages

N turbine rotor speed (rev/min)

P static pressure (newtons/cm², psia)

P_T total pressure (newtons/cm², psia)

R_N Reynolds number

S suction surface length (cm, in.)

TED trailing edge diameter (cm, in.)

TER trailing edge radius (cm, in.)

T_T total temperature (°K, °R)

U wheel speed (m/sec, ft/sec)

V air velocity (m/sec, ft/sec)

W turbine airflow (kg/sec, lbm/sec)

SYMBOLS AND DEFINITIONS (Cont'd)

au	airfoil pitch (cm, in.)
λ	airfoil throat (cm, in.)
ρ	density (kg/m ³ , lbm/ft ³)
σ	airfoil solidity, ratio of chord to spacing
α	absolute air angle, degrees
β	relative air angle, degrees
$\theta_{ m r}$	ratio of air temperature to standard sea level temperature
θ	turning through an airfoil
φ2	ratio of the square of the airfoil absolute exit velocity to the ideal exit velocity
μ	viscosity (kg/sec-m, lbm/sec ft)
Subscripts	
m	mean
o	vane inlet
1	vane exit
1.5	blade inlet
2	blade exit
v	vane
В	blade
A	absolute reference
R	relative reference
Superscripts	
*	metal angles

SYMBOLS AND DEFINITIONS (Cont'd)

Definitions

$$\alpha$$
 gaging = $\arcsin \lambda/\tau$

$$\mathbf{D_f} \qquad = \frac{1 - \cos \beta_1}{\cos \beta_2} + \frac{\cos \beta_1 (\tan \beta_1 + \tan \beta_2)}{2\sigma}$$

$$R_N = \frac{\rho V S}{\mu}$$

Reaction =
$$(P_1 - P_2)/(P_0 - P_2)$$

Uncovered = turning angle depicted by Γ in Figure 6 Turning

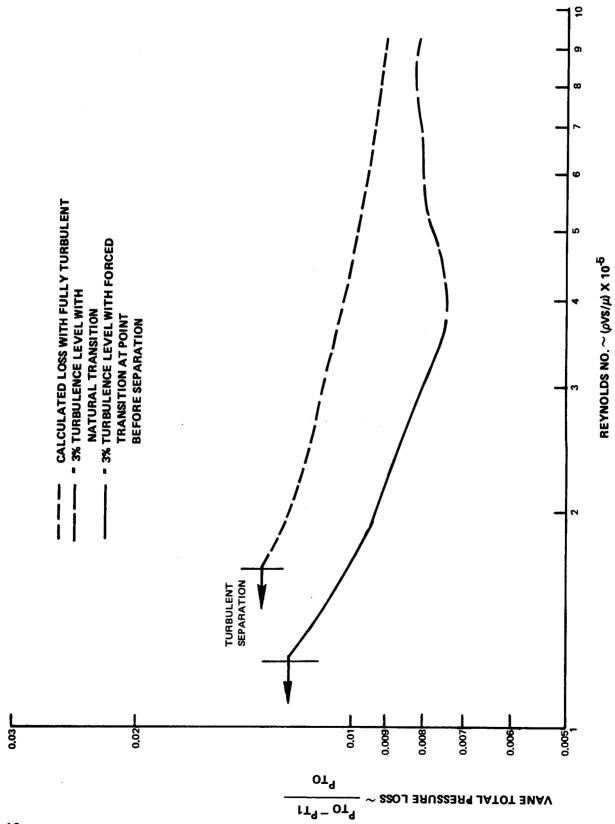


FIGURE 1 CASCADE PROFILE LOSS VS REYNOLDS NUMBER

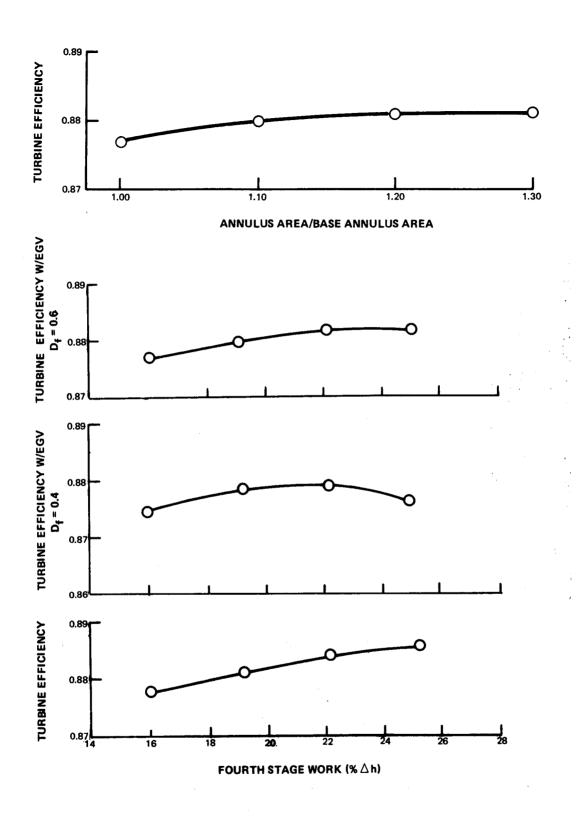
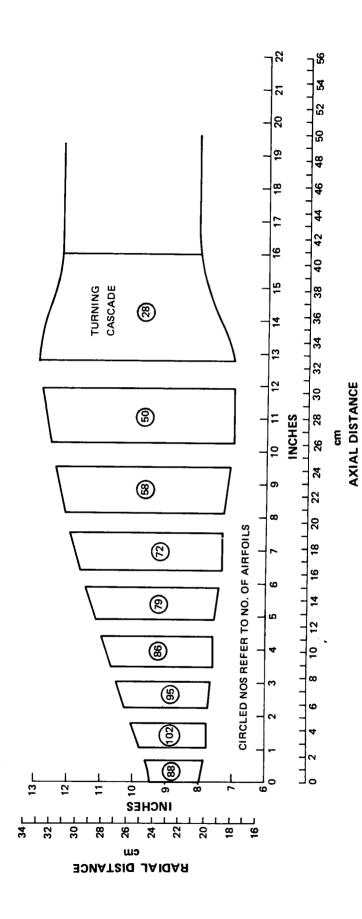


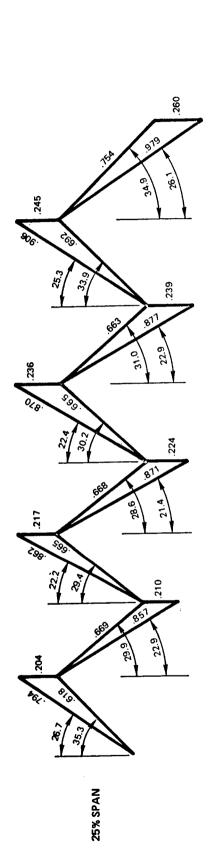
FIGURE 2 TURBINE EFFICIENCY VS. FOURTH STAGE
WORK AND EXIT ANNULUS AREA



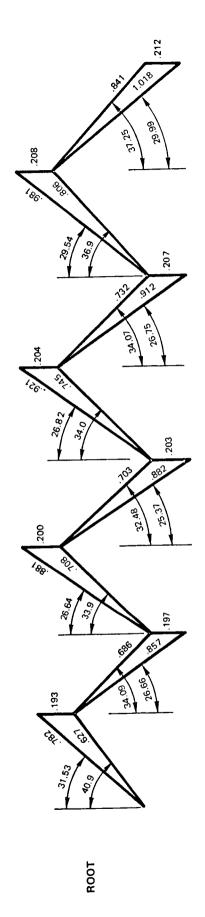
TURBINE FLOWPATH

FIGURE 3

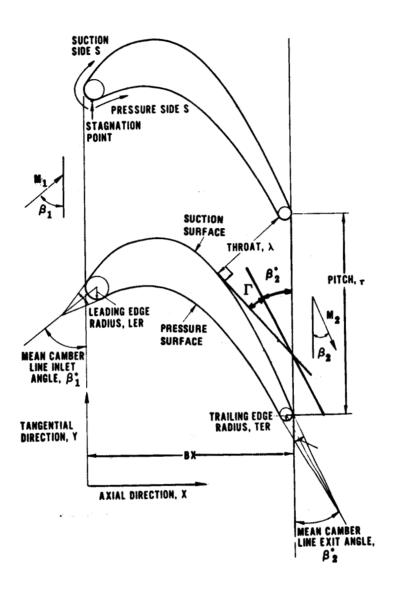
14



NUMBERS SHOWN ARE ANGLES IN DEGREES AND MACH NUMBERS AT THE AIRFOIL EXIT



NUMBERS SHOWN ARE ANGLES IN DEGREES AND MACH NUMBERS AT THE AIRFOIL EXIT FIGURE 5 VELOCITY DIAGRAMS



NOMENCLATURE IN FIGURE IS FOR A BLADE. FOR A VANE REPLACE " β " to " α "

FIGURE 6 AIRFOIL NOMENCLATURE

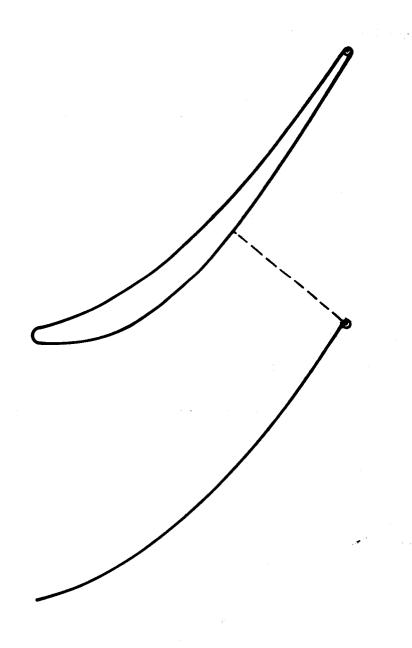


FIG. 7 FIRST STAGE VANE ROOT 5.0 SCALE

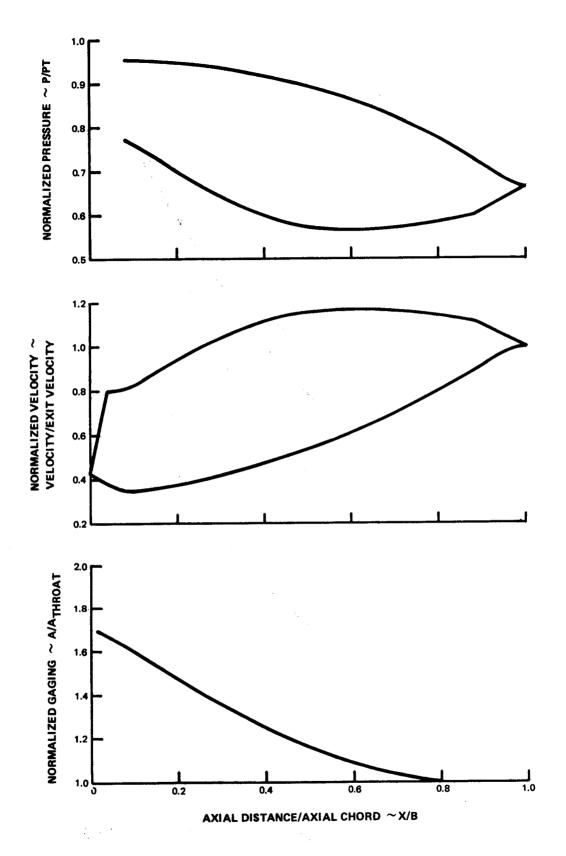


FIG. 8 FIRST STAGE VANE ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

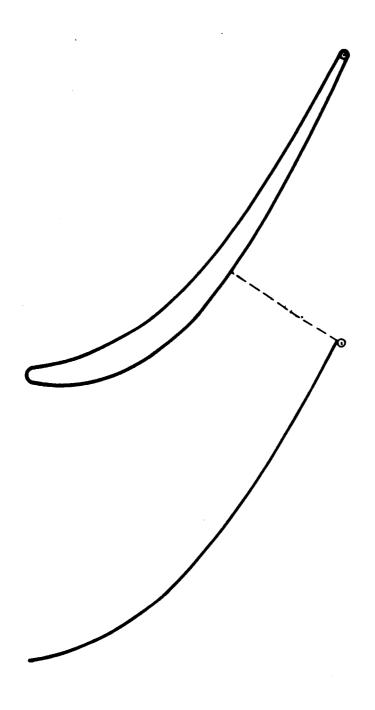


FIG. 9 FIRST STAGE VANE QUARTER ROOT 5.0 SCALE

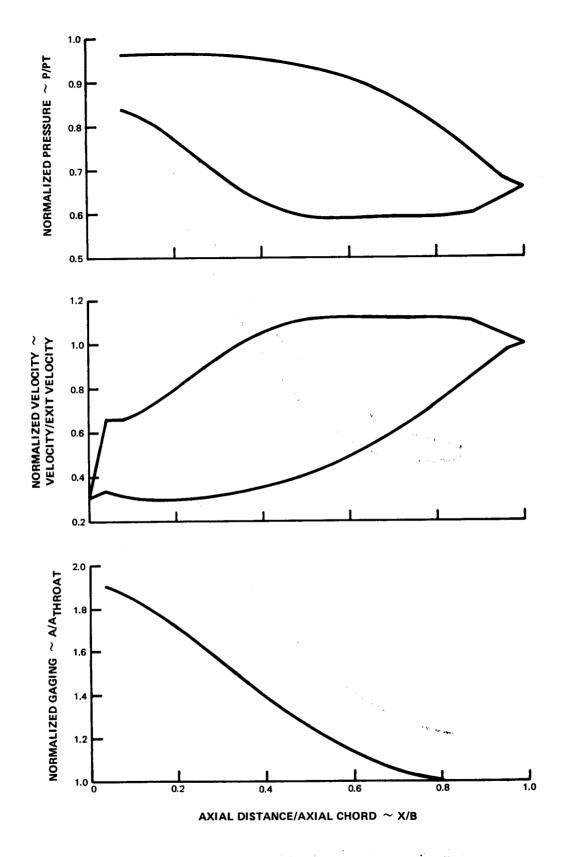


FIG. 10 FIRST STAGE VANE QUARTER ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

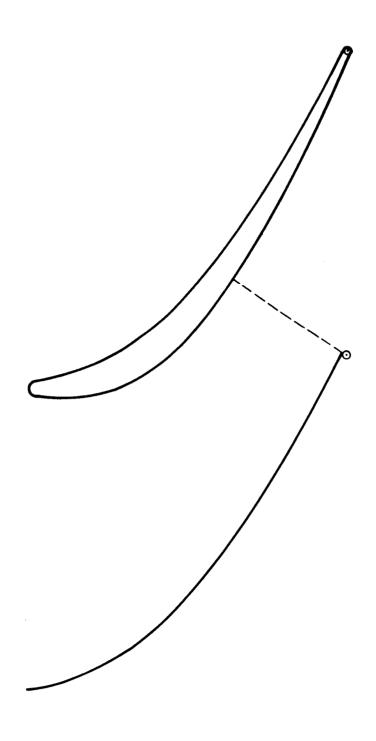


FIG. 11 FIRST STAGE VANE MEAN 5.0 SCALE

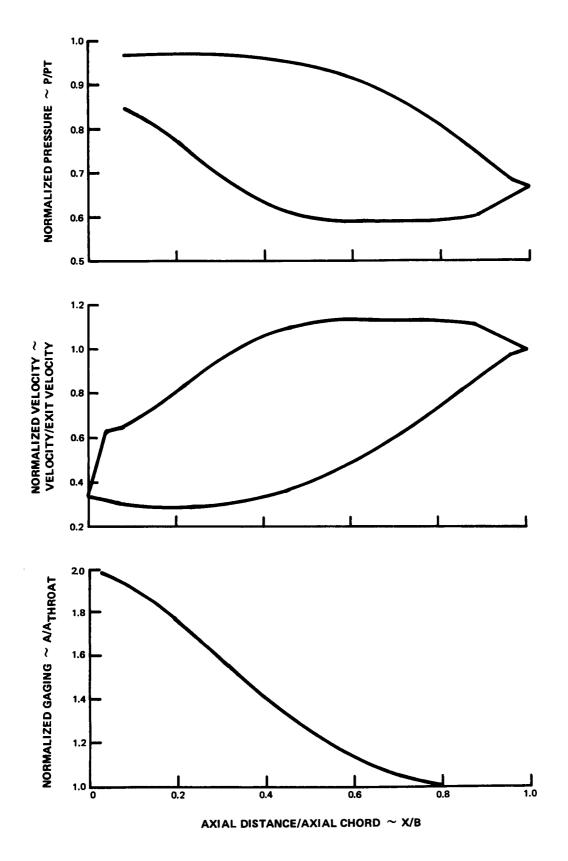


FIG. 12 FIRST STAGE VANE MEAN NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

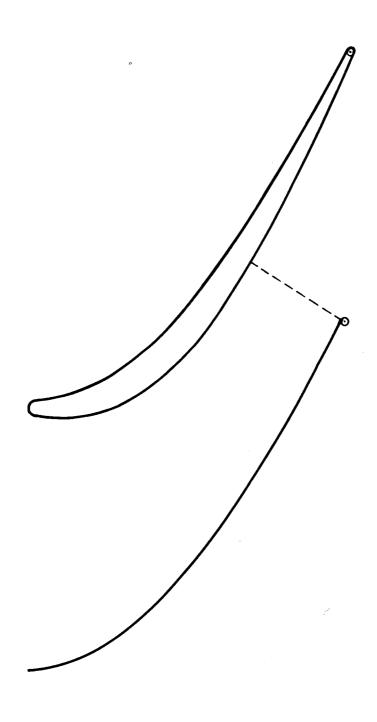


FIG. 13 FIRST STAGE VANE QUARTER TIP 5.0 SCALE

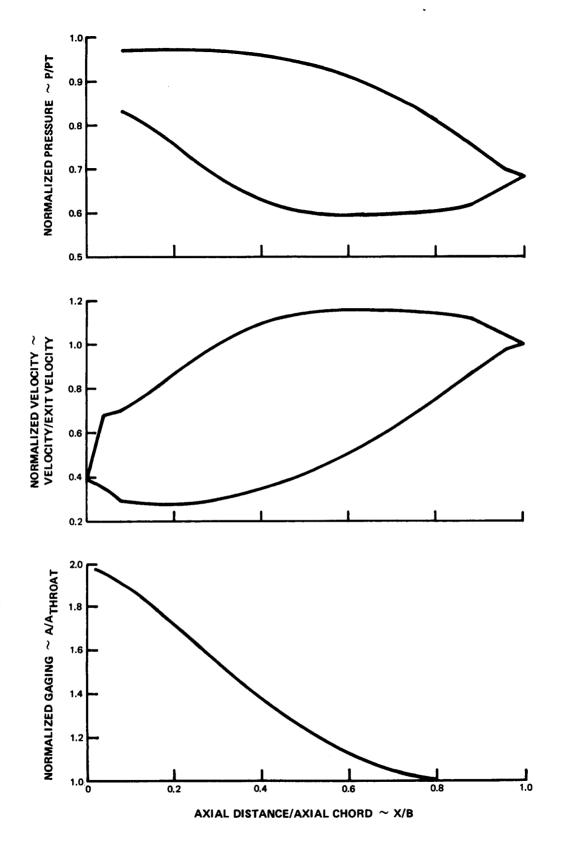


FIG. 14 FIRST STAGE VANE QUARTER TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

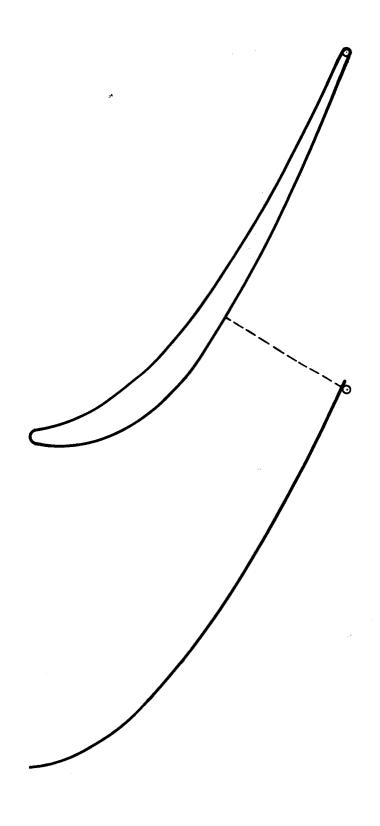


FIG. 15 FIRST STAGE VANE TIP

5.0 SCALE

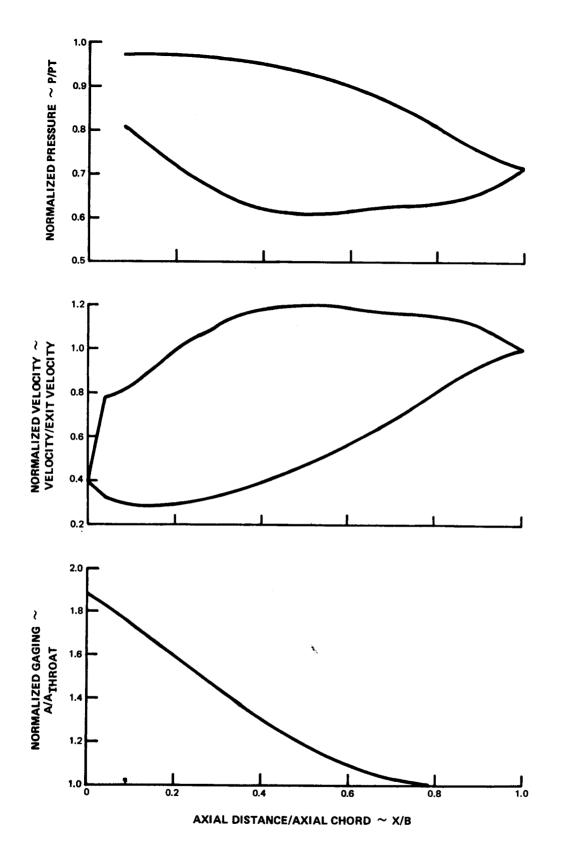


FIG. 16 FIRST STAGE VANE TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

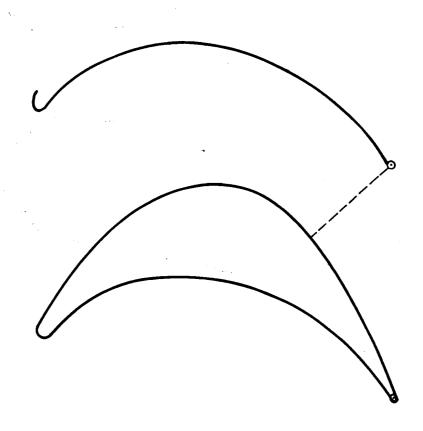


FIG. 17 FIRST STAGE BLADE ROOT 5.0 SCALE

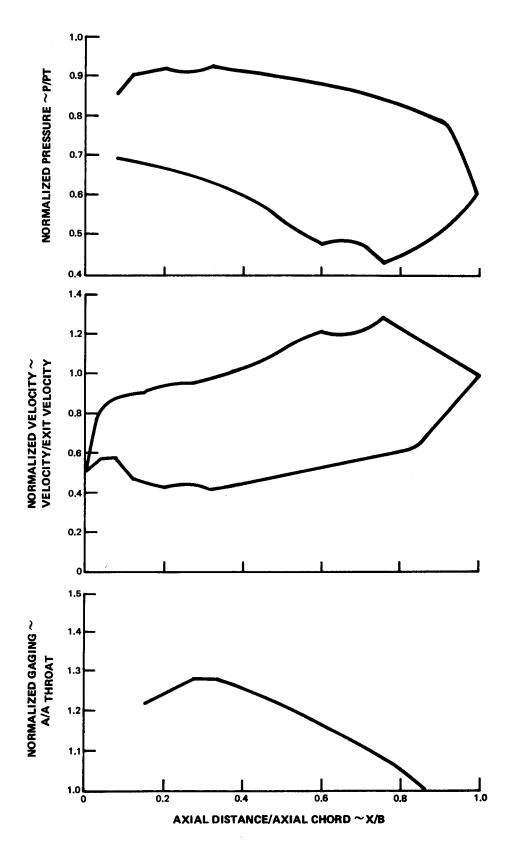


FIG. 18 FIRST STAGE BLADE ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

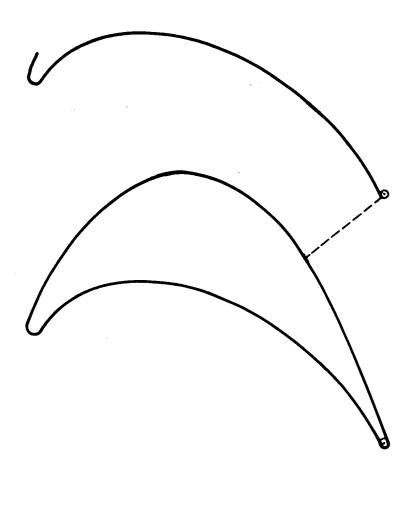


FIG. 19 FIRST STAGE BLADE QUARTER ROOT 5.0 SCALE

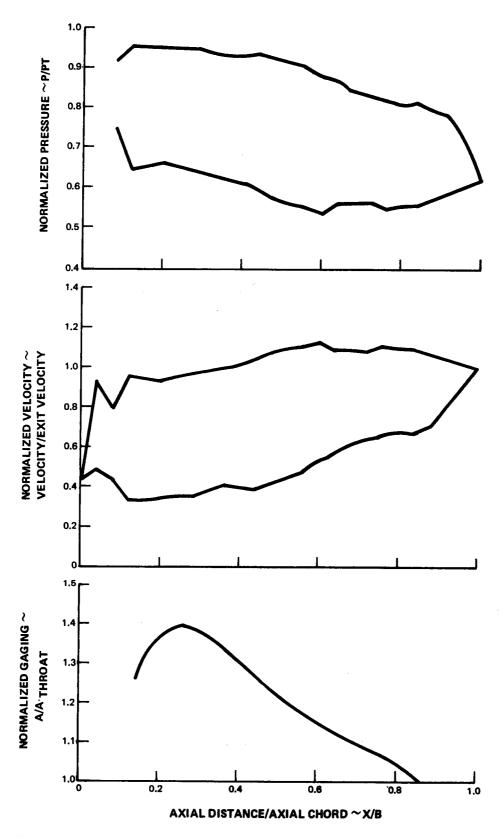


FIG. 20 FIRST STAGE BLADE QUARTER ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

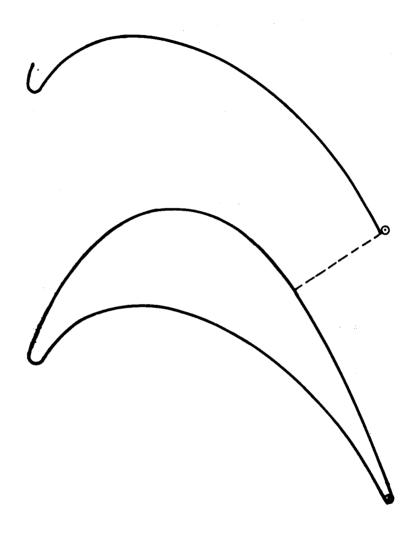


FIG. 21 FIRST STAGE BLADE MEAN 5.0 SCALE

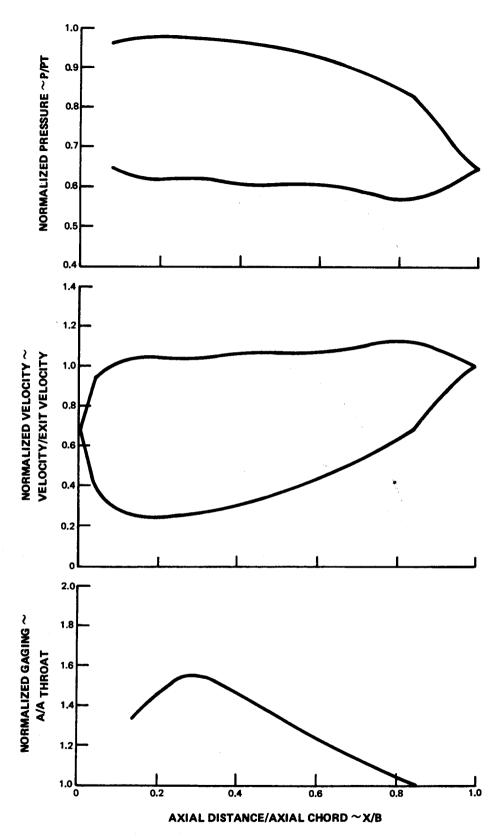


FIG. 22 FIRST STAGE BLADE MEAN NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

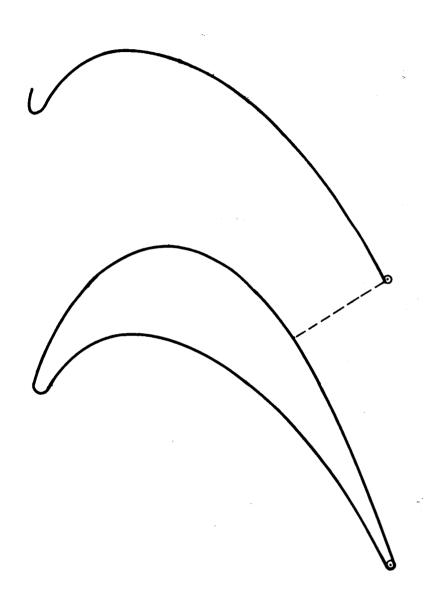


FIG. 23 FIRST STAGE BLADE QUARTER TIP 5.0 SCALE

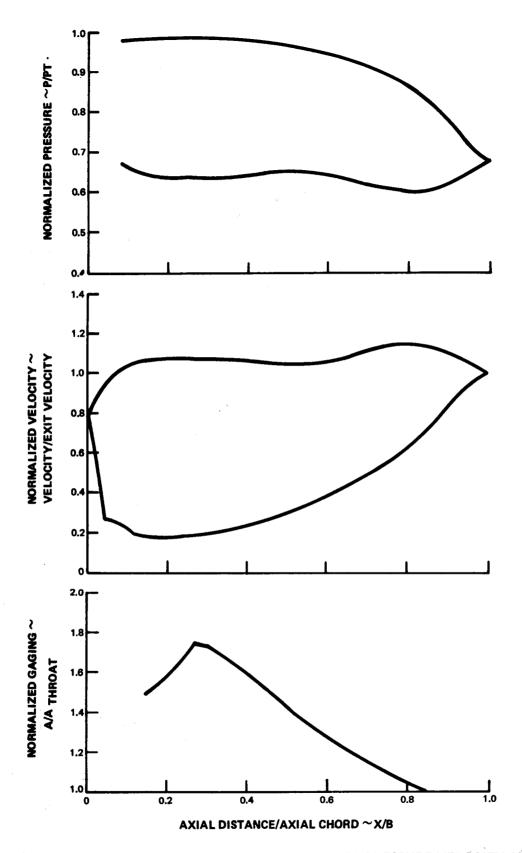


FIG. 24 FIRST STAGE BLADE QUARTER TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

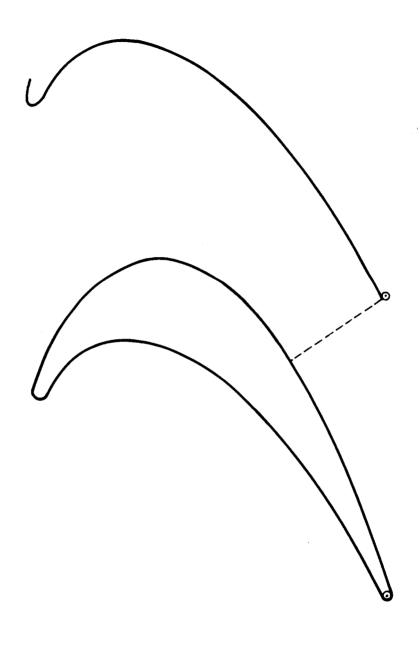


FIG. 25 FIRST STAGE BLADE TIP 5.0 SCALE

FIG. 27 SECOND STAGE VANE ROOT (5.0 SCALE)

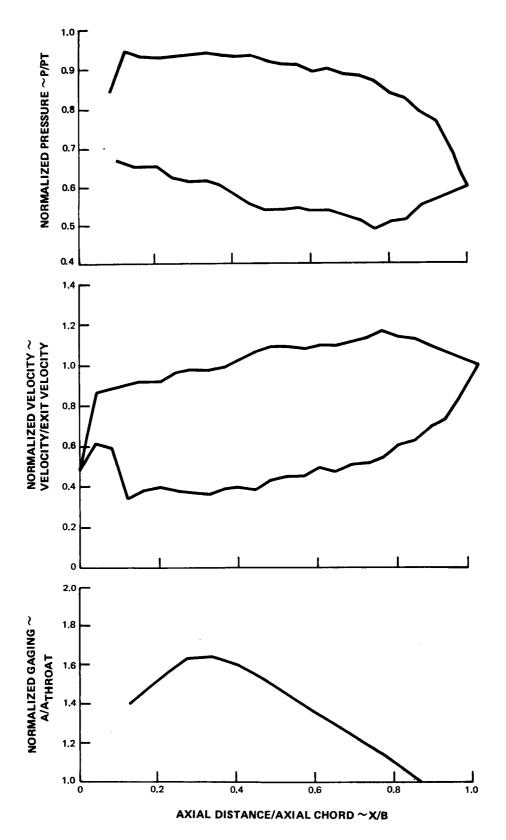


FIG. 28 SECOND STAGE VANE ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

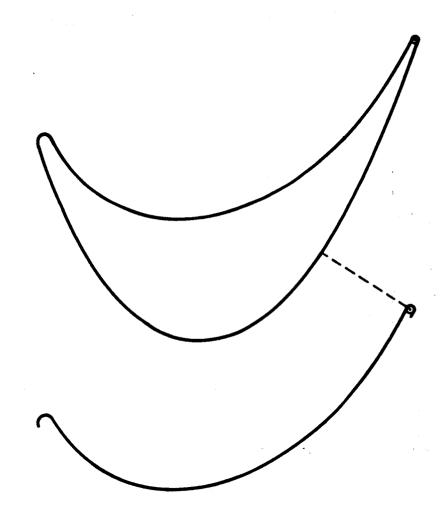


FIG. 29 SECOND STAGE VANE QUARTER ROOT (5.0 SCALE)

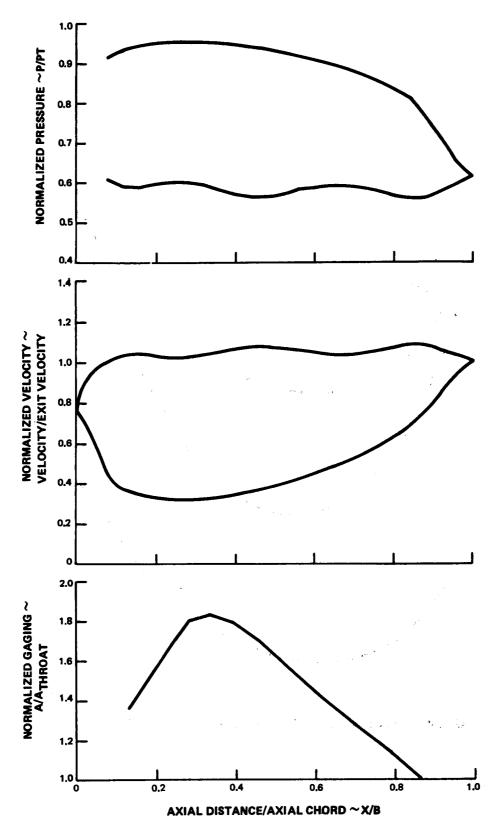


FIG.30 SECOND STAGE VANE QUARTER ROOT NORMALIZED PRESEURE VELOCITY AND GAGING DIAGRAMS

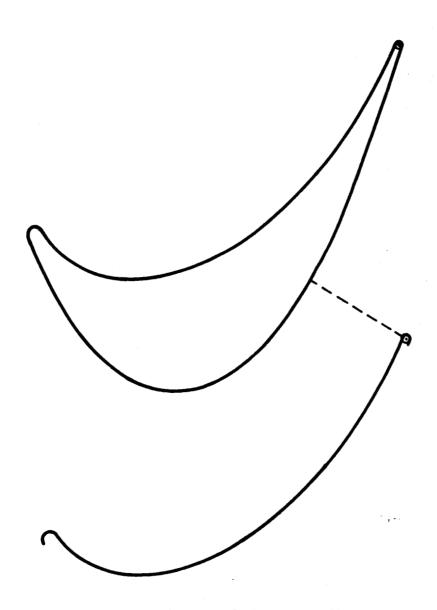


FIG. 31 SECOND STAGE VANE MEAN (5.0 SCALE)

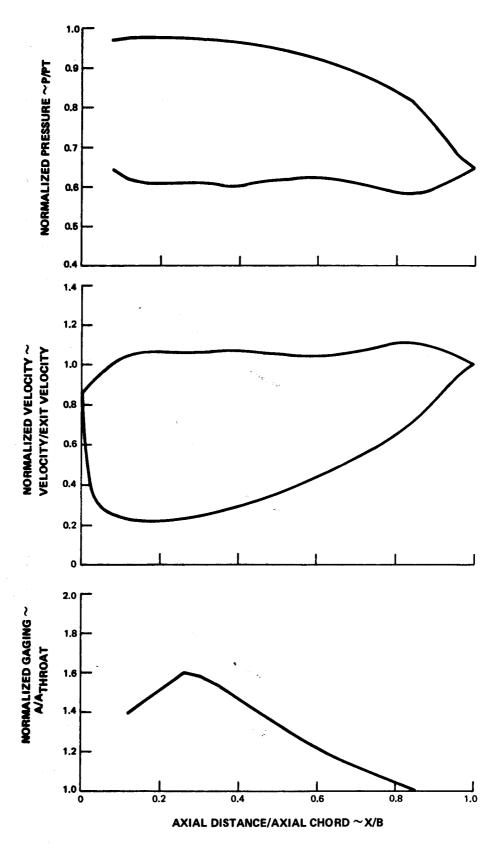


FIG. 32 SECOND STAGE VANE MEAN NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

FIG. 33 SECOND STAGE VANE QUARTER TIP (5.0 SCALE)

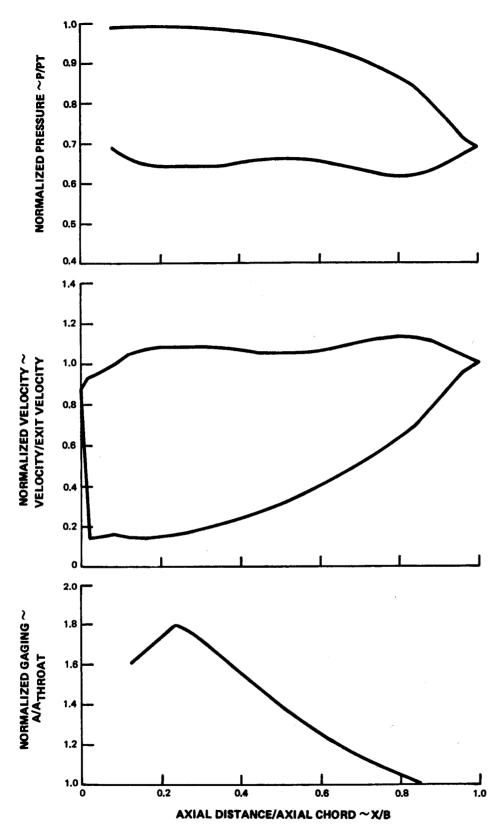


FIG. 34 SECOND STAGE VANE QUARTER TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

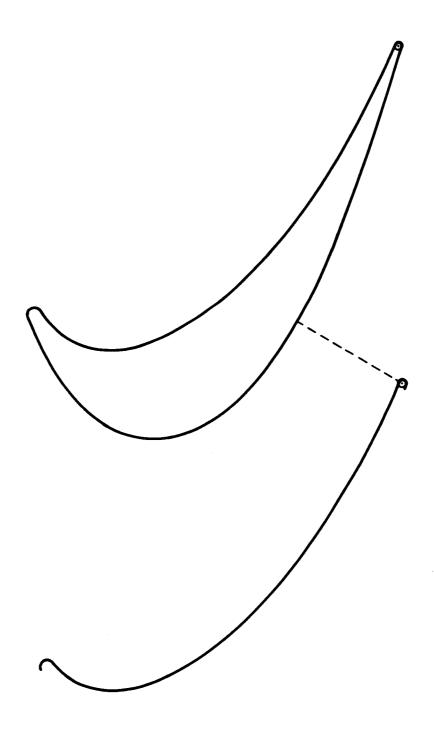


FIG. 35 SECOND STAGE VANE TIP (5.0 SCALE)

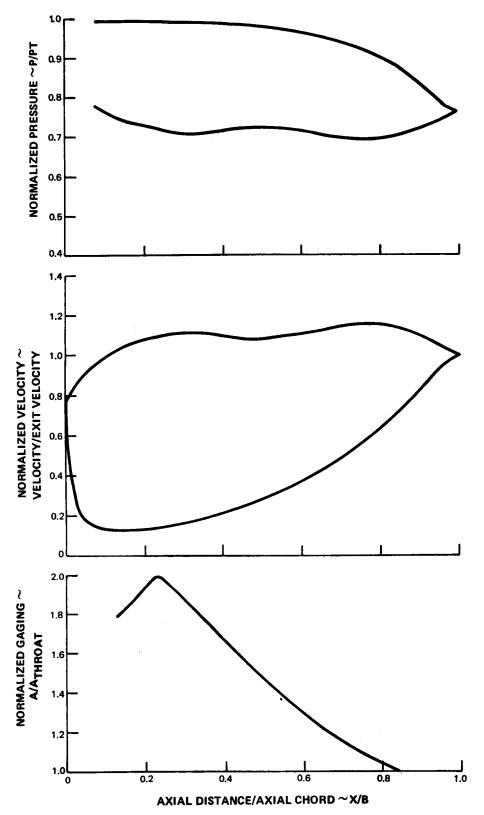


FIG. 36 SECOND STAGE VANE TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

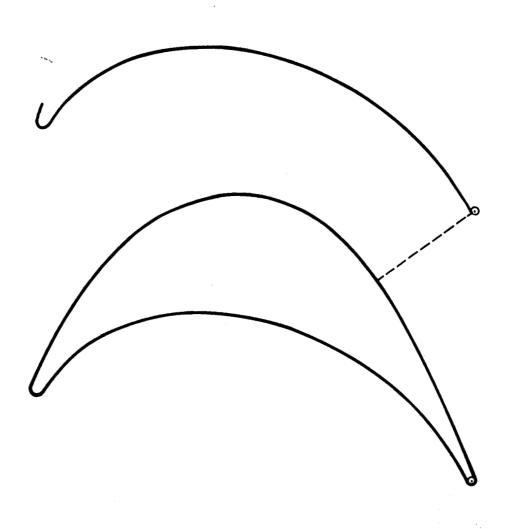


FIG. 37 SECOND STAGE BLADE ROOT 5.0 SCALE

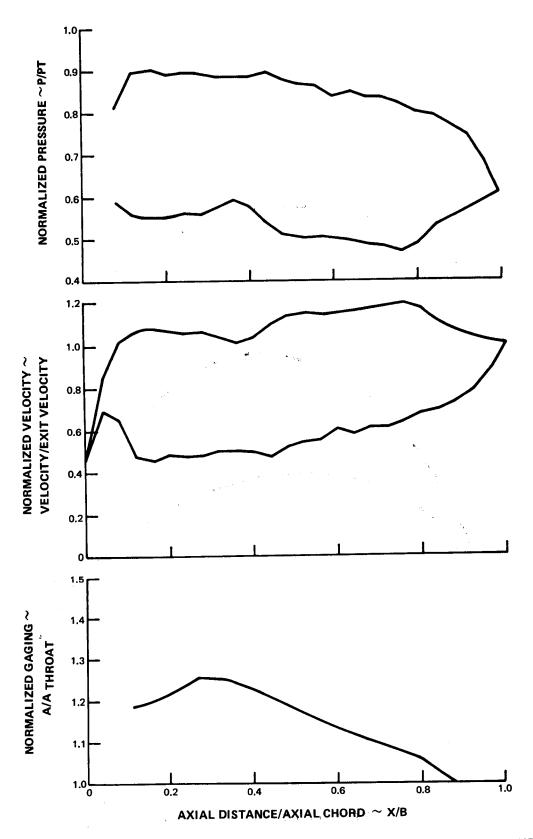


FIG. 38 SECOND STAGE BLADE ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

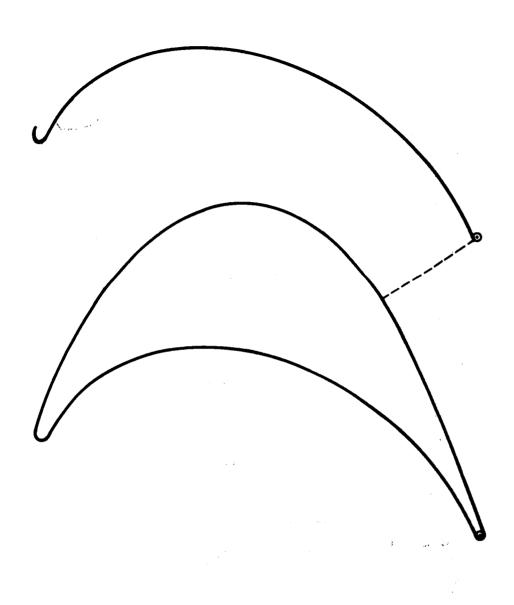


FIG. 39 SECOND STAGE BLADE QUARTER ROOT 5.0 SCALE

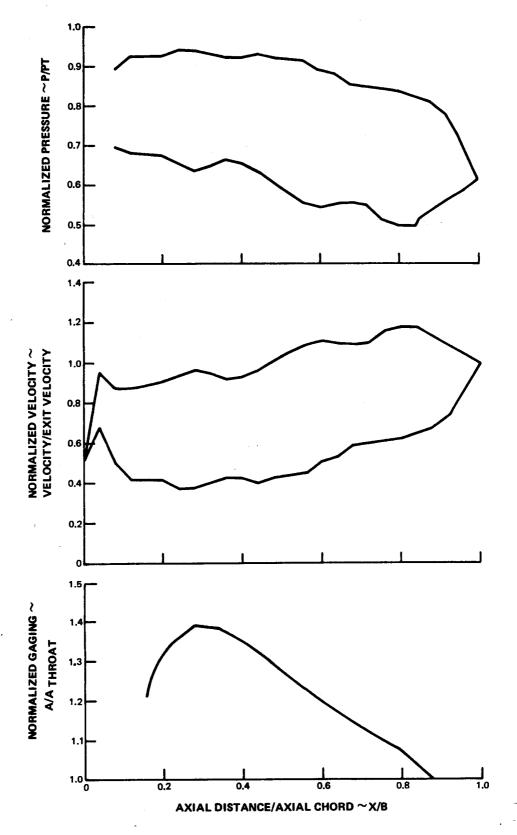


FIG. 40 SECOND STAGE BLADE QUARTER ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

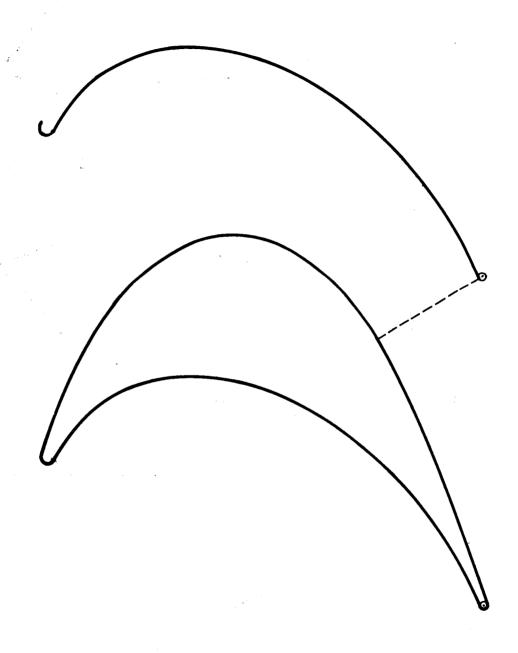


FIG. 41 SECOND STAGE BLADE MEAN 5.0 SCALE

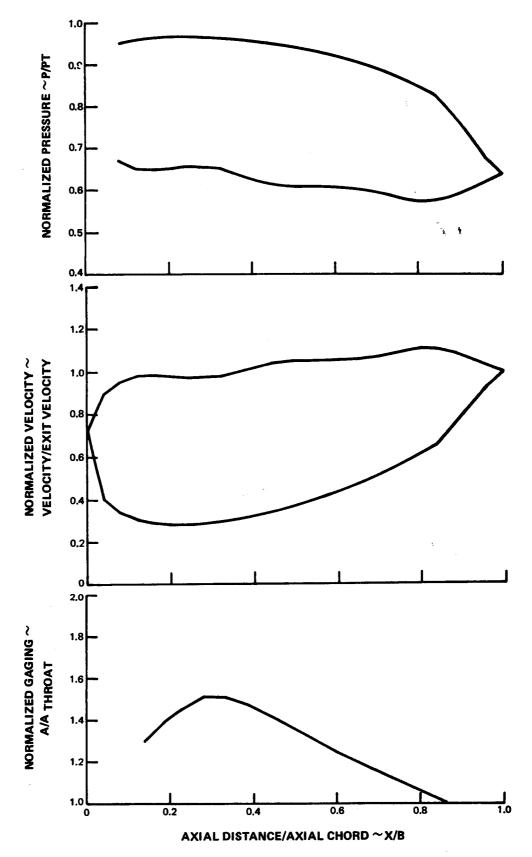


FIG. 42 SECOND STAGE BLADE MEAN NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

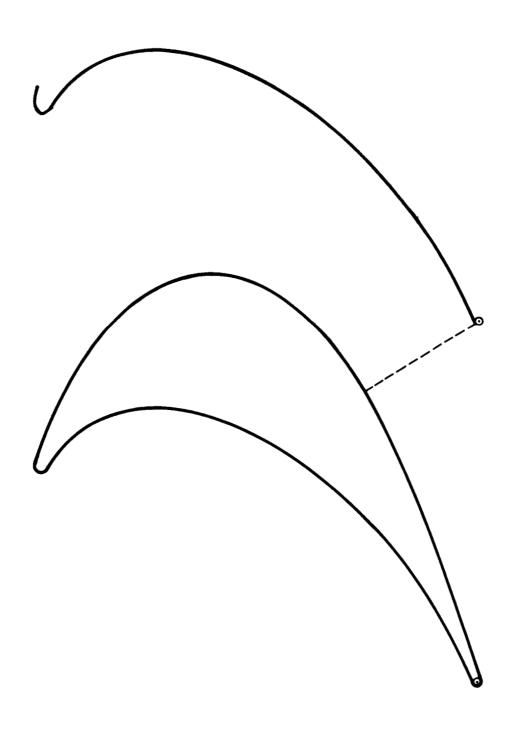


FIG. 43 SECOND STAGE BLADE QUARTER TIP 5.0 SCALE

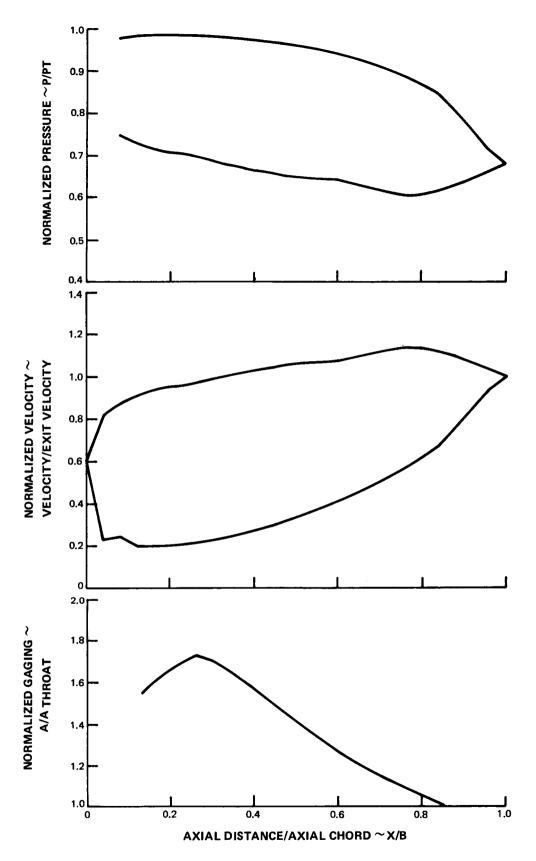


FIG. 44 SECOND STAGE BLADE QUARTER TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

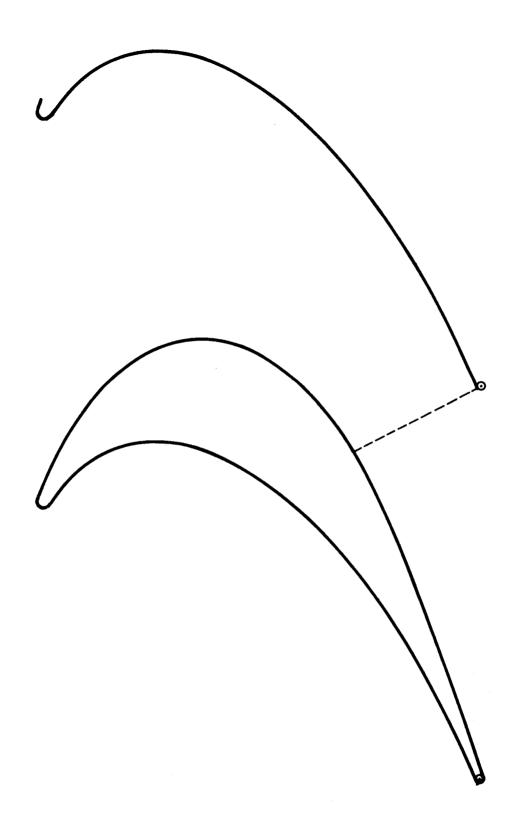


FIG. 45 SECOND STAGE BLADE TIP

5.0 SCALE

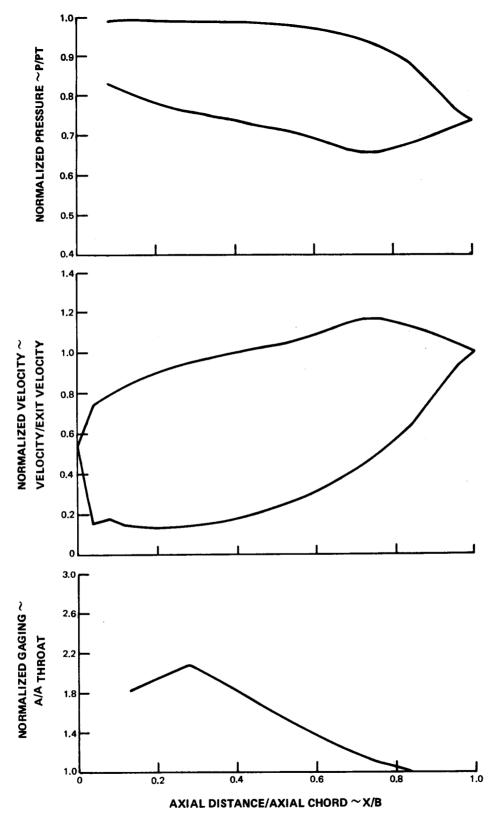


FIG. 46 SECOND STAGE BLADE TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

FIG. 47 THIRD STAGE VANE ROOT (5.0 SCALE)

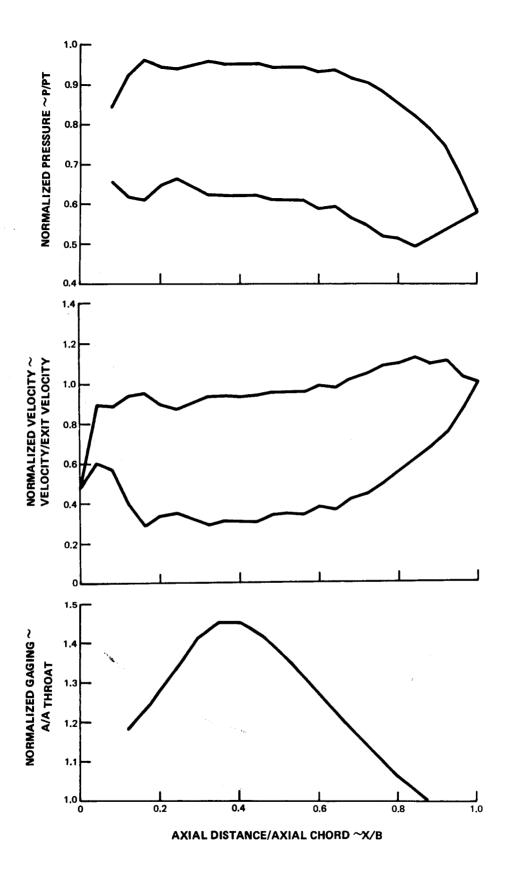


FIG. 48 THIRD STAGE VANE ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

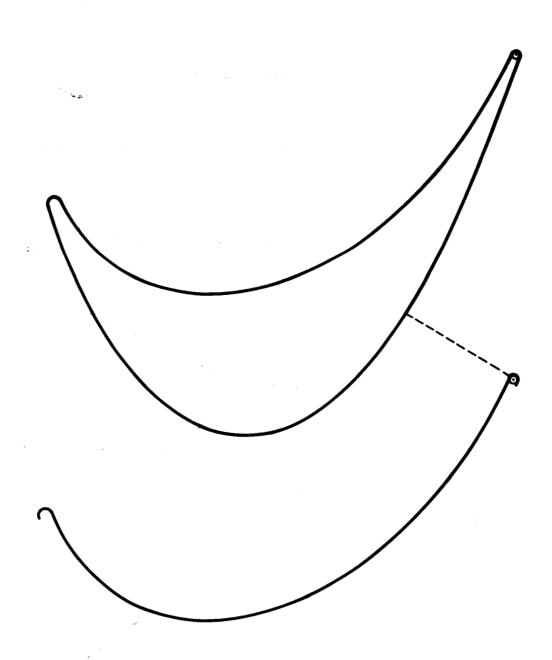


FIG. 49 THIRD STAGE VANE QUARTER ROOT (5.0 SCALE)

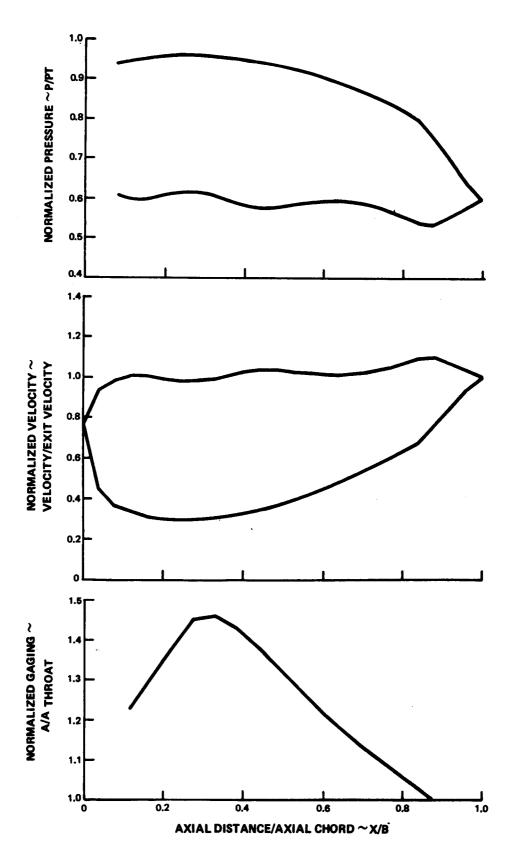


FIG. 50 THIRD STAGE VANE QUARTER ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

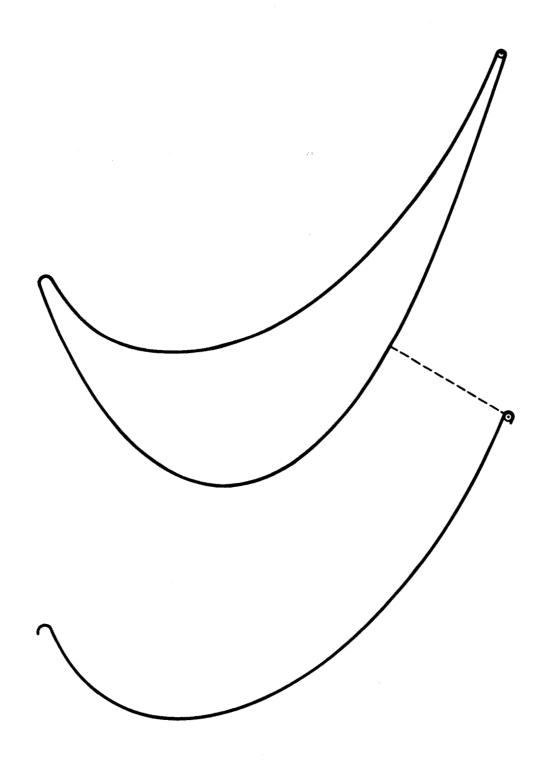


FIG. 51 THIRD STAGE VANE MEAN (5.0 SCALE)

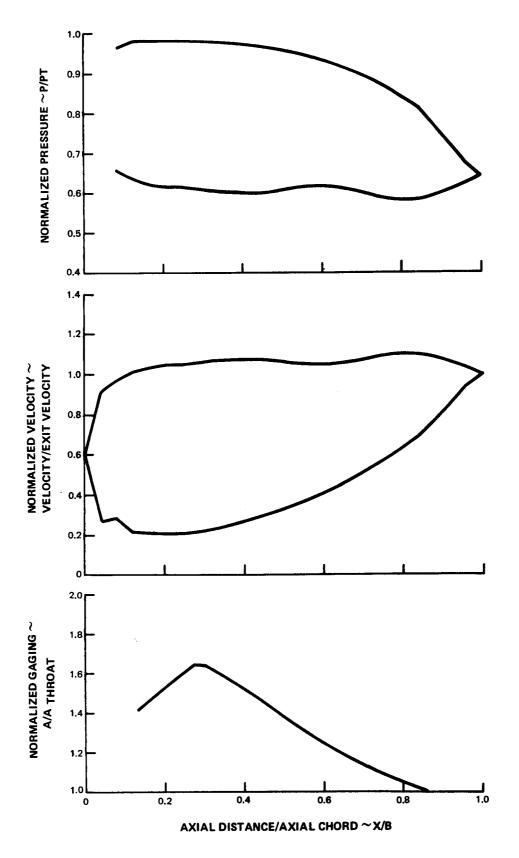


FIG. 52 THIRD STAGE VANE MEAN NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

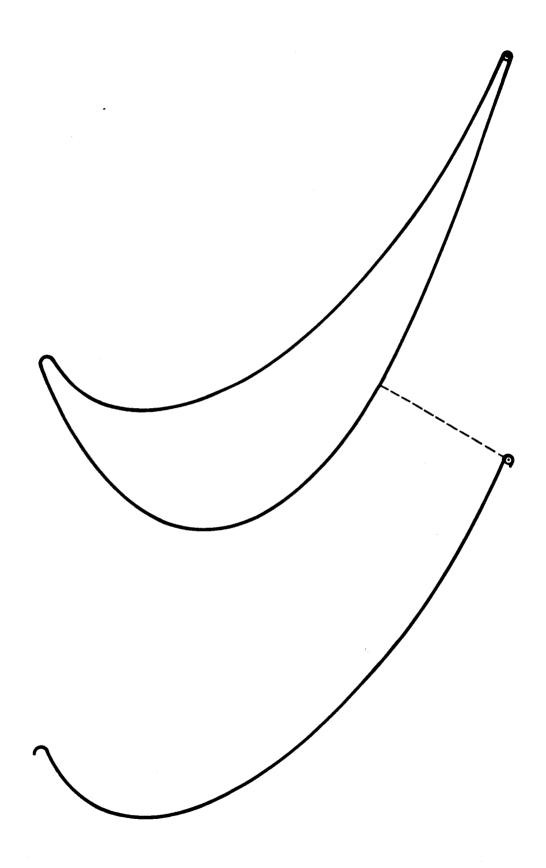


FIG. 53 THIRD STAGE VANE QUARTER TIP (5.0 SCALE)

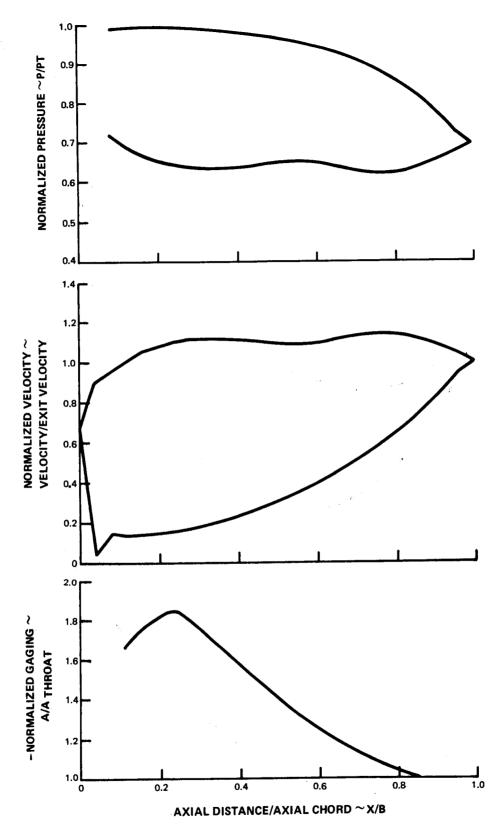


FIG. 54 THIRD STAGE VANE QUARTER TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

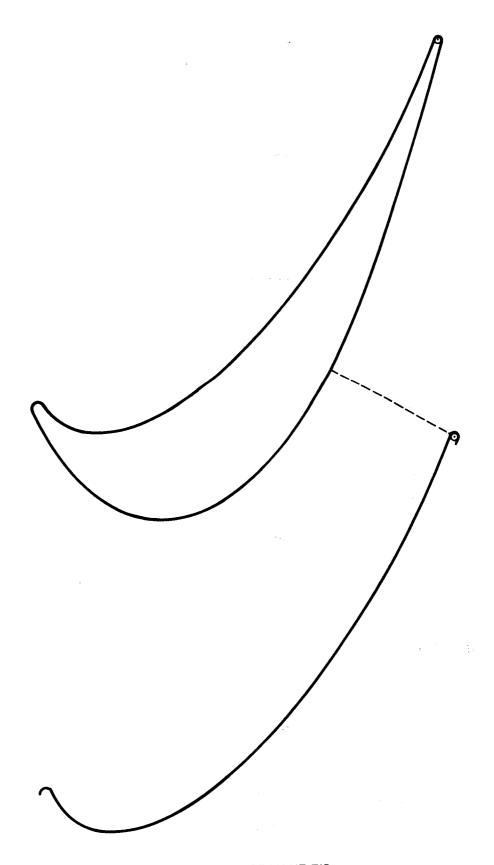


FIG. 55 THIRD STAGE VANE TIP (5.0 SCALE)

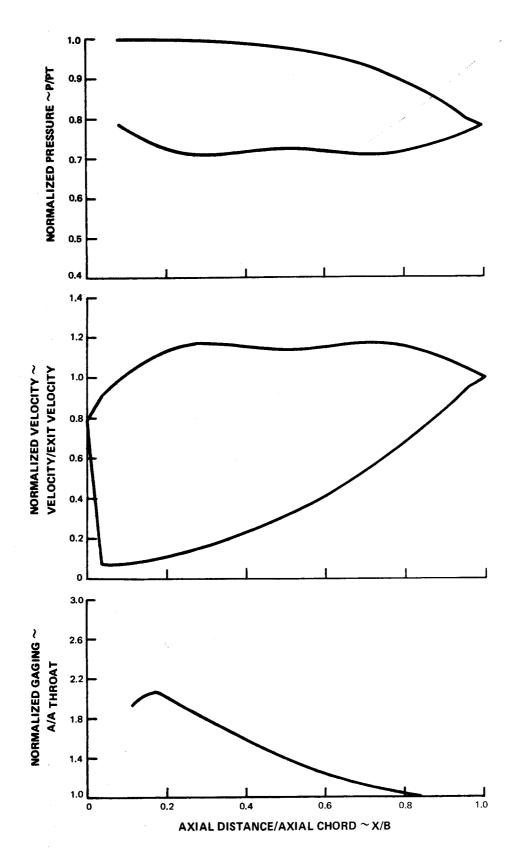


FIG. 56 THIRD STAGE VANE TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

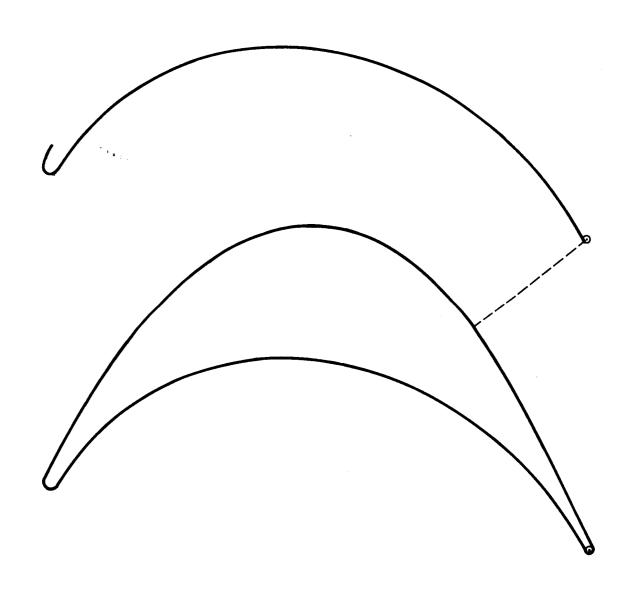


FIG. 57 THIRD STAGE BLADE ROOT 5.0 SCALE

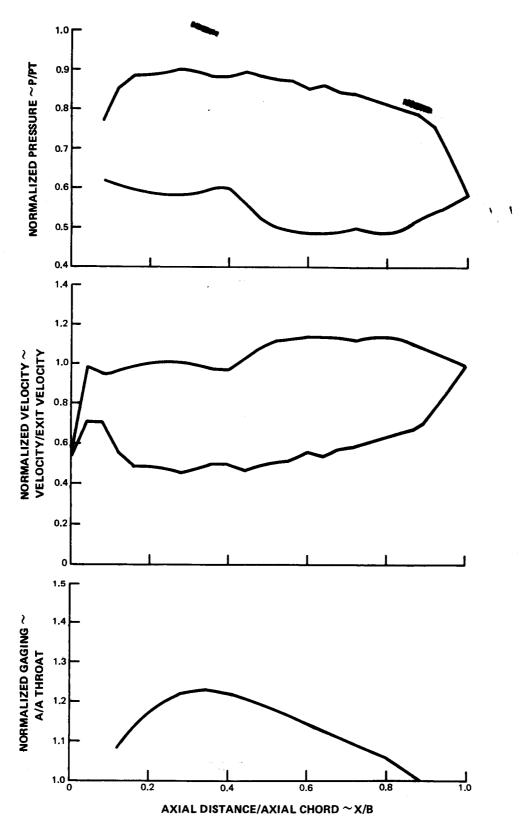


FIG. 58 THIRD STAGE BLADE ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

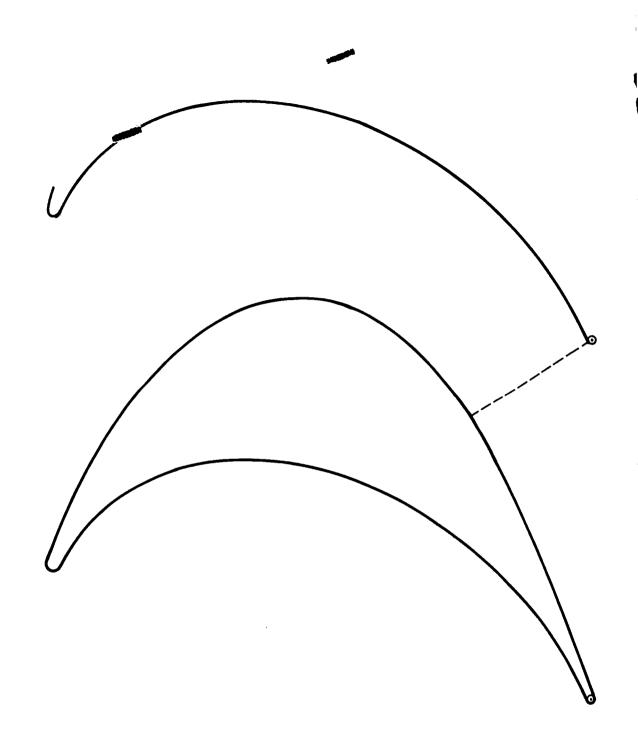


FIG. 59 THIRD STAGE BLADE QUARTER ROOT (5.0 SCALE)

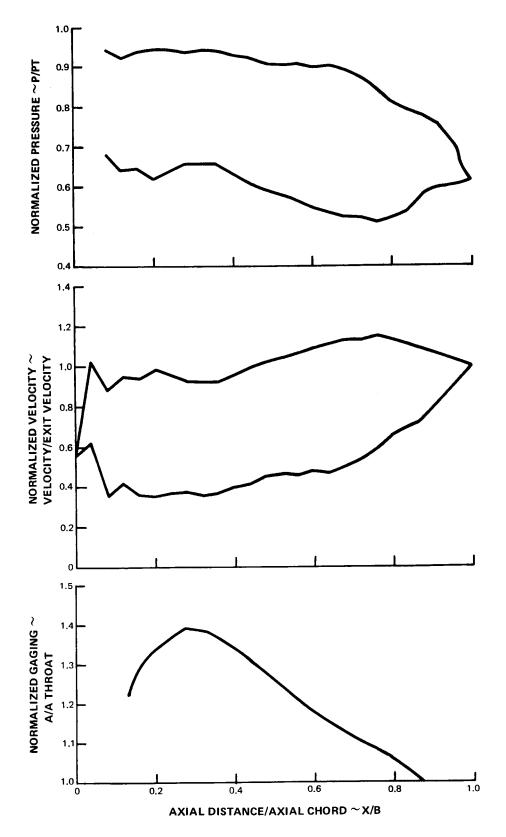


FIG. 60 THIRD STAGE BLADE QUARTER ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

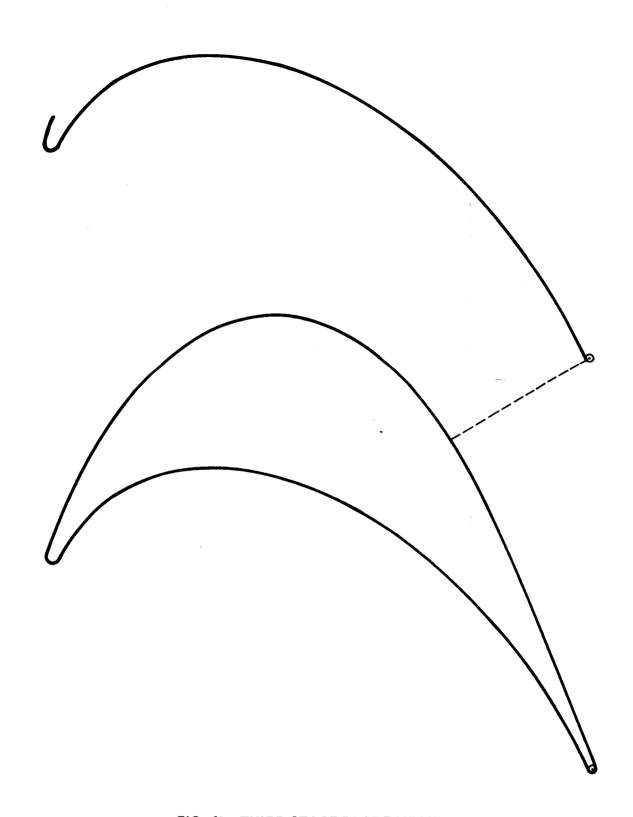


FIG. 61 THIRD STAGE BLADE MEAN 5.0 SCALE

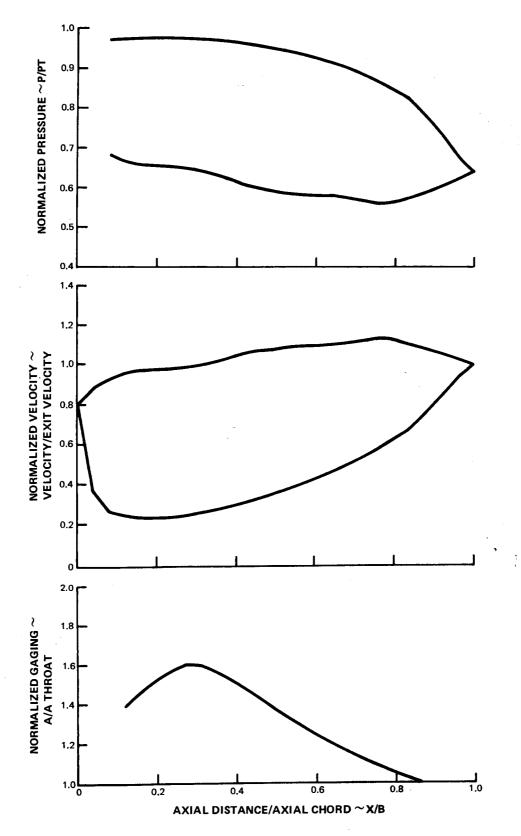


FIG. **62** THIRD STAGE BLADE MEAN NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

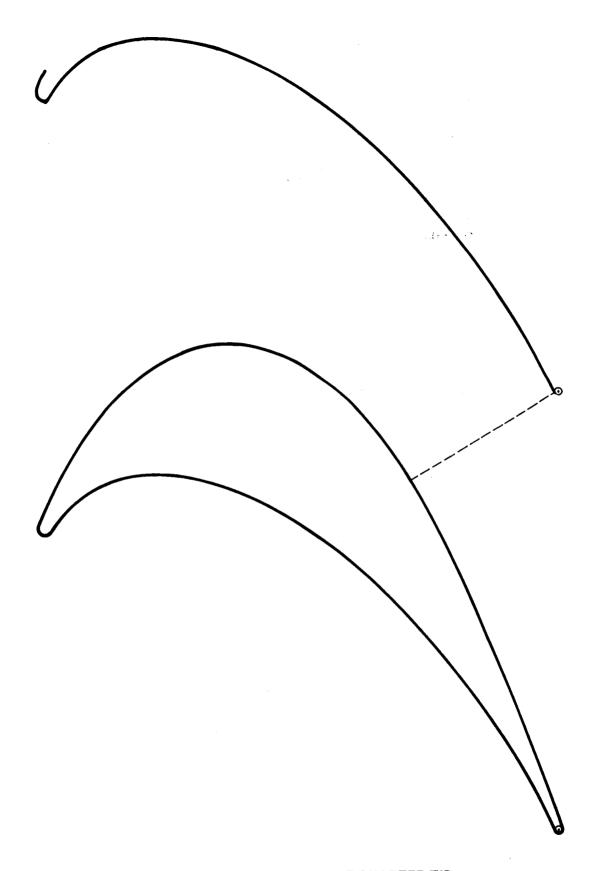


FIG. 63 THIRD STAGE BLADE QUARTER TIP

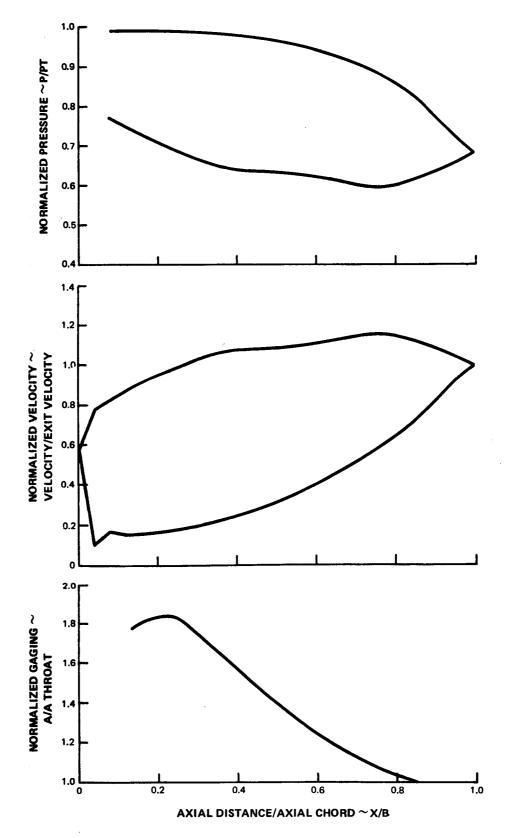
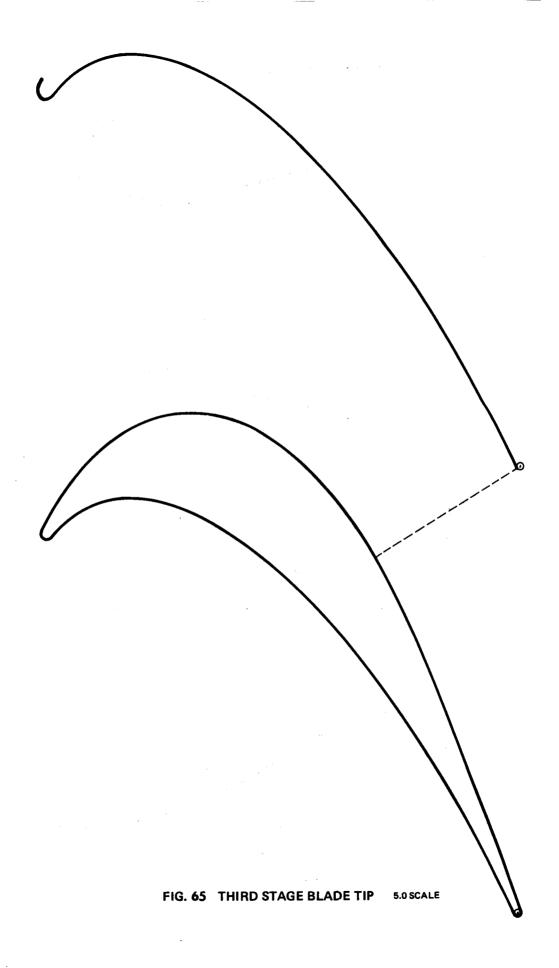


FIG. 64 THIRD STAGE BLADE QUARTER TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS



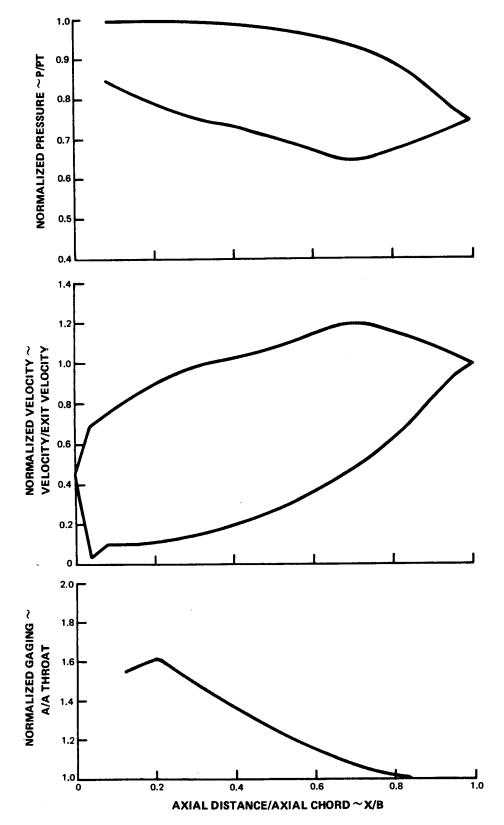


FIG. 66 THIRD STAGE BLADE TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

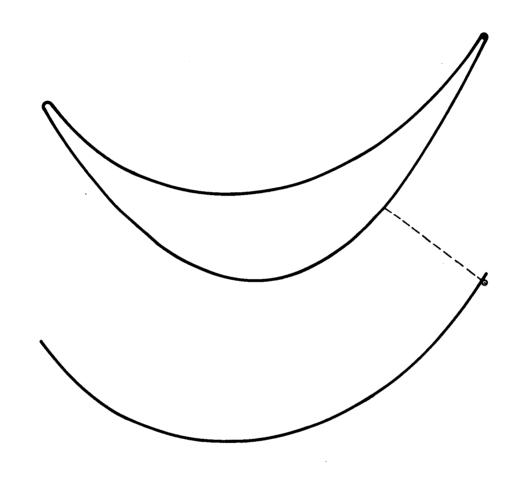


FIG. 67 FOURTH STAGE VANE ROOT (2.5 SCALE)

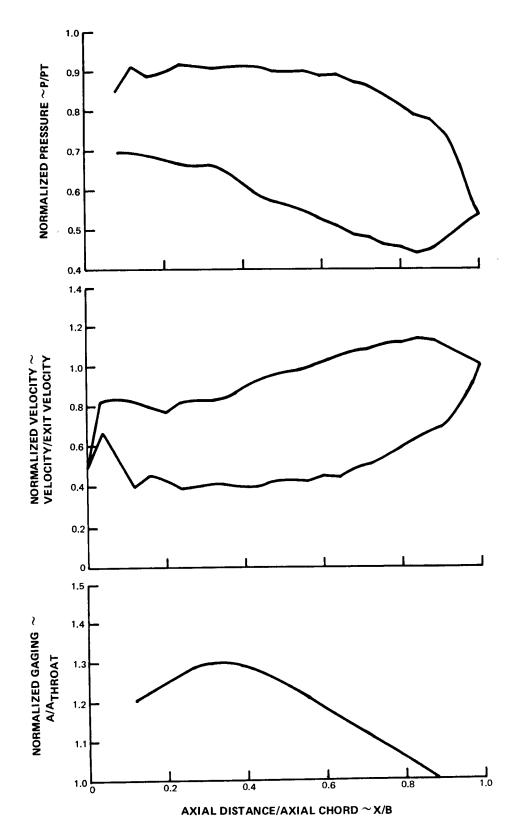


FIG. 68 FOURTH STAGE VANE ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

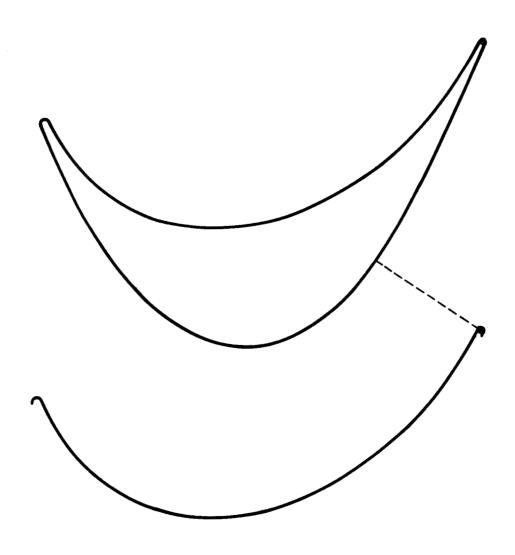


FIG. 69 FOURTH STAGE VANE QUARTER ROOT (2.5 SCALE)

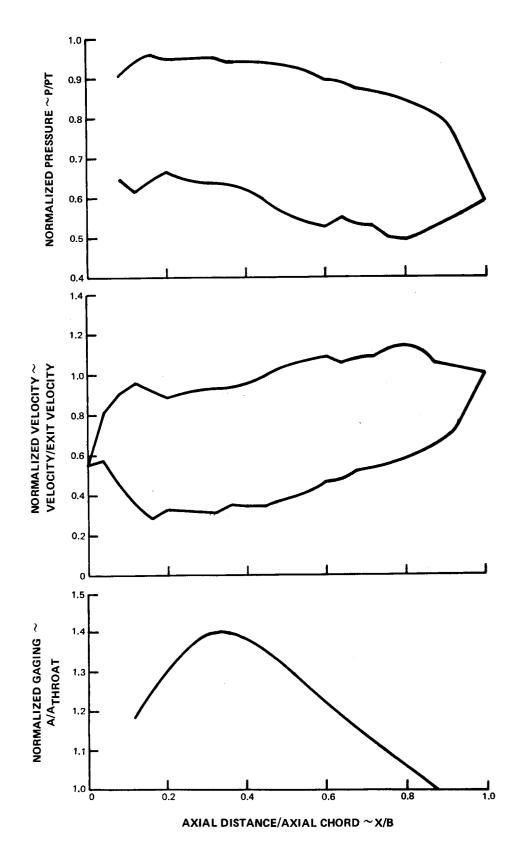


FIG. 70 FOURTH STAGE VANE QUARTER ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

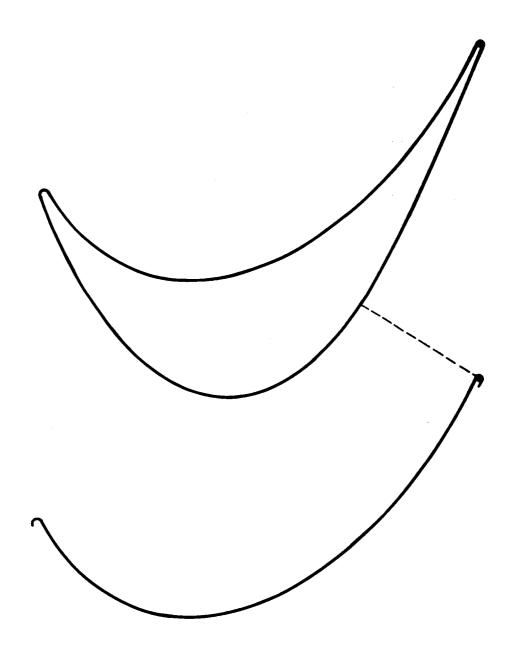


FIG. 71 FOURTH STAGE VANE MEAN

(2.5 SCALE)

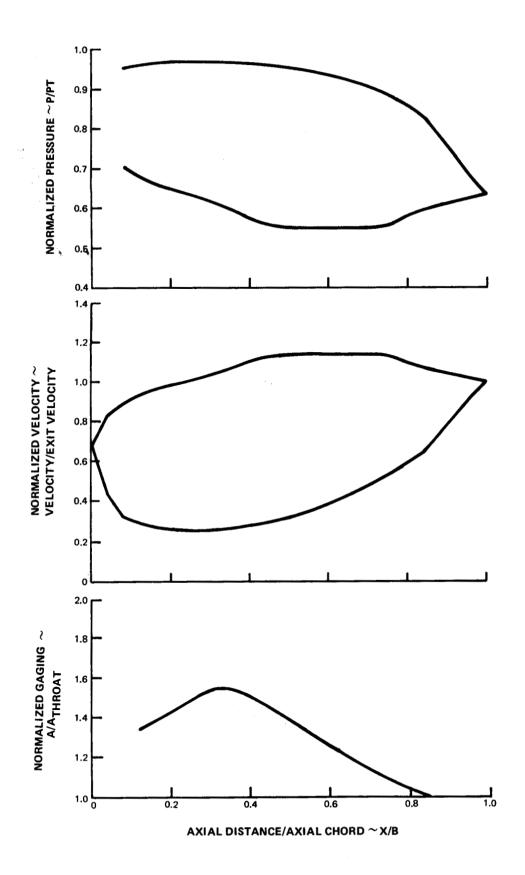


FIG. 72 FOURTH STAGE VANE MEAN NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

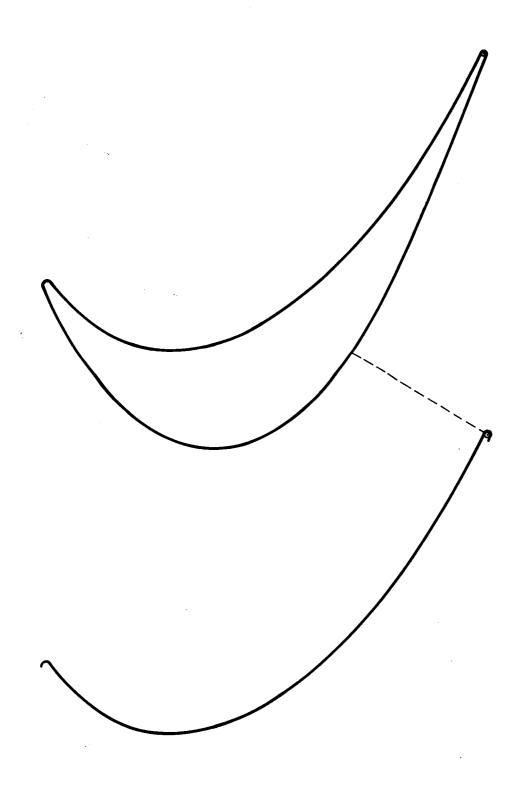


FIG. 73 FOURTH STAGE VANE QUARTER TIP (2.5 SCALE)

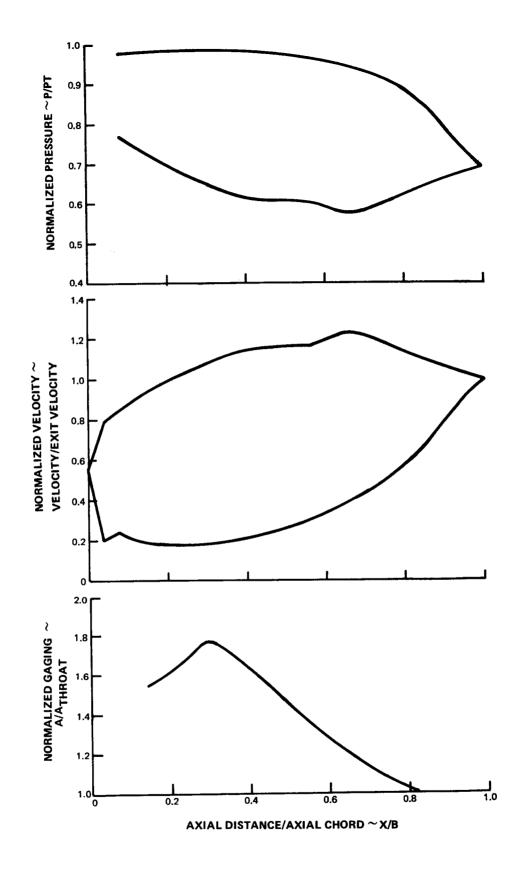


FIG. 74 FOURTH STAGE VANE QUARTER TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

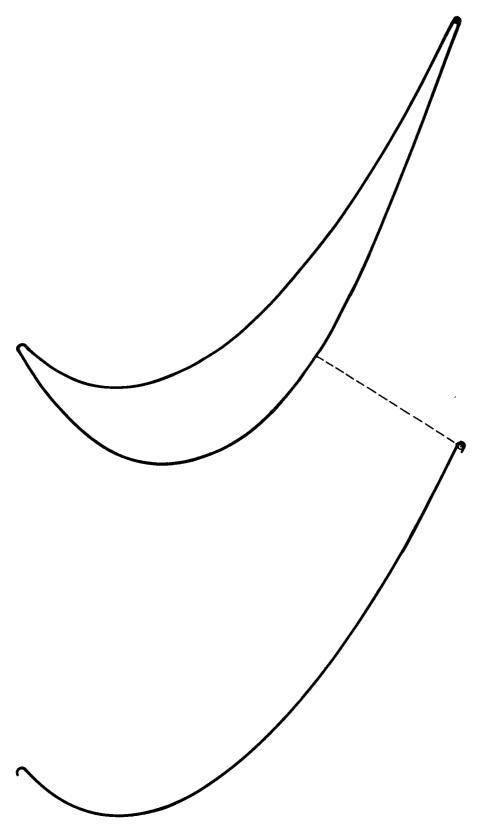


FIG. 75 FOURTH STAGE VANE TIP (2.5 SCALE)

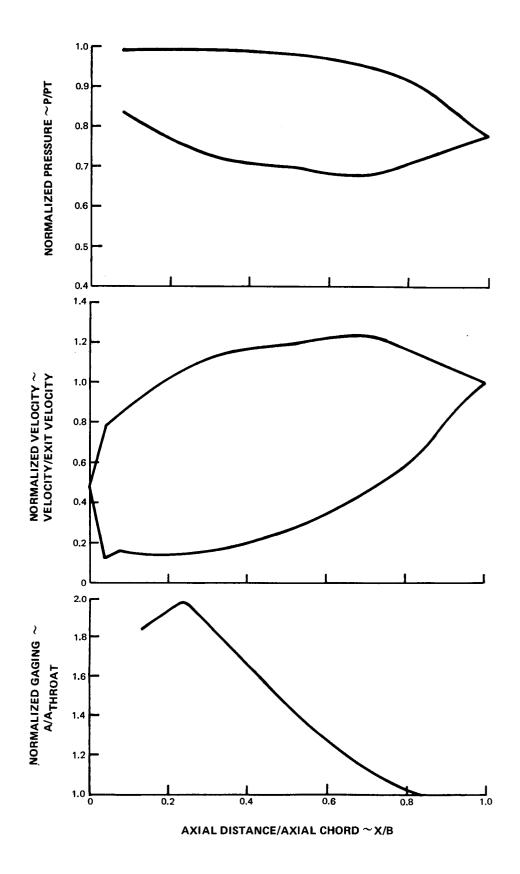


FIG. **76** FOURTH STAGE VANE TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

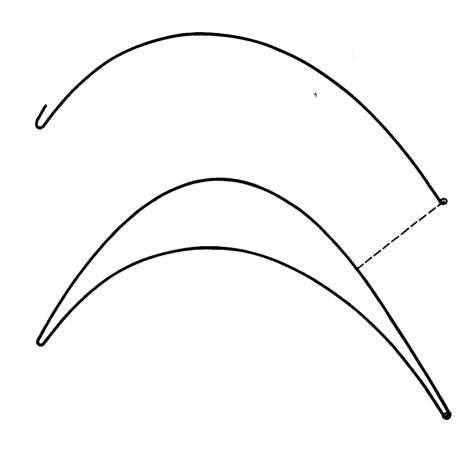


FIG. 77 FOURTH STAGE BLADE ROOT 2.5 SCALE

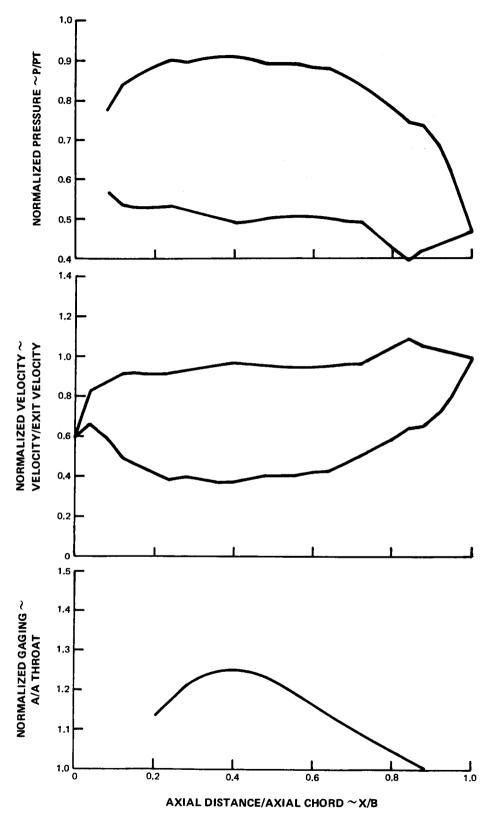


FIG. 78 FOURTH STAGE BLADE ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

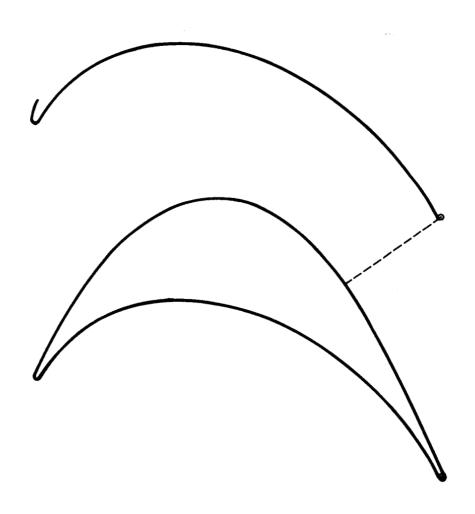


FIG. 79 FOURTH STAGE BLADE QUARTER ROOT

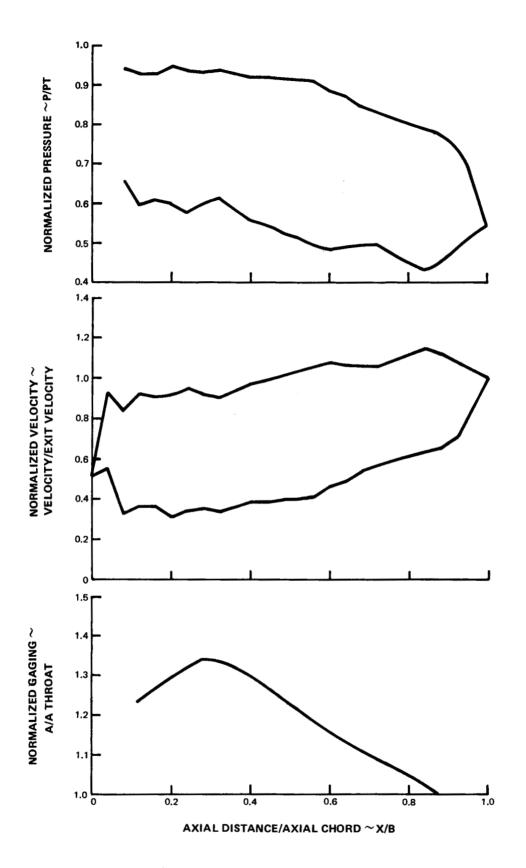


FIG. 80 FOURTH STAGE BLADE QUARTER ROOT NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

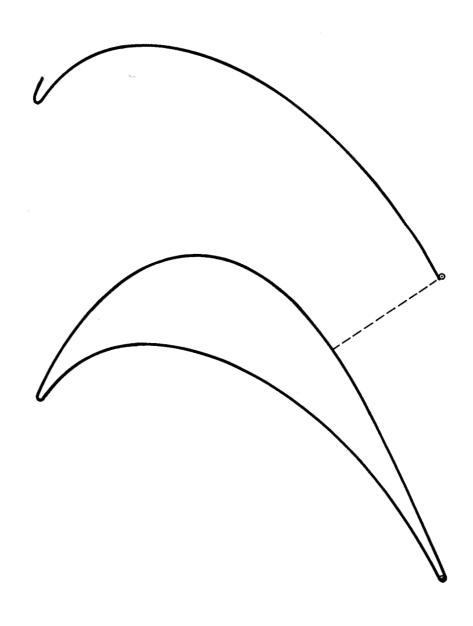


FIG. 81 FOURTH STAGE BLADE MEAN 2.5 SCALE

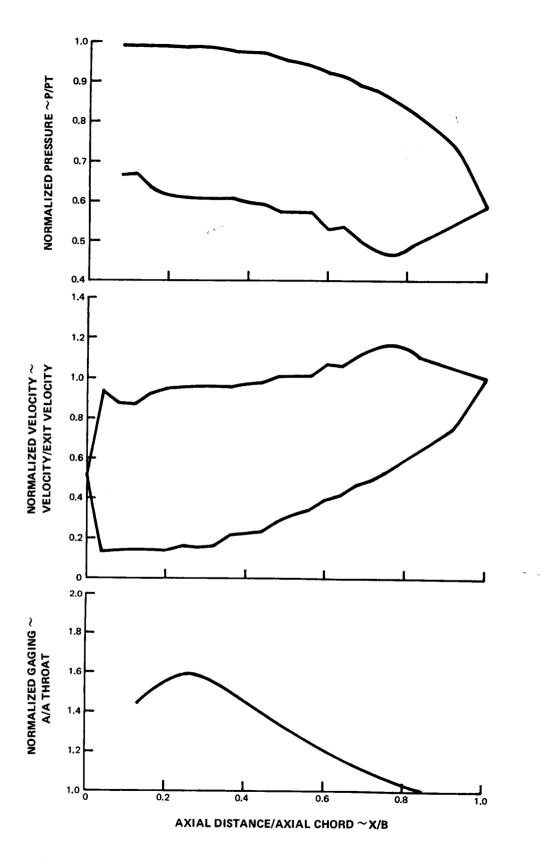


FIG. 82 FOURTH STAGE BLADE MEAN NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

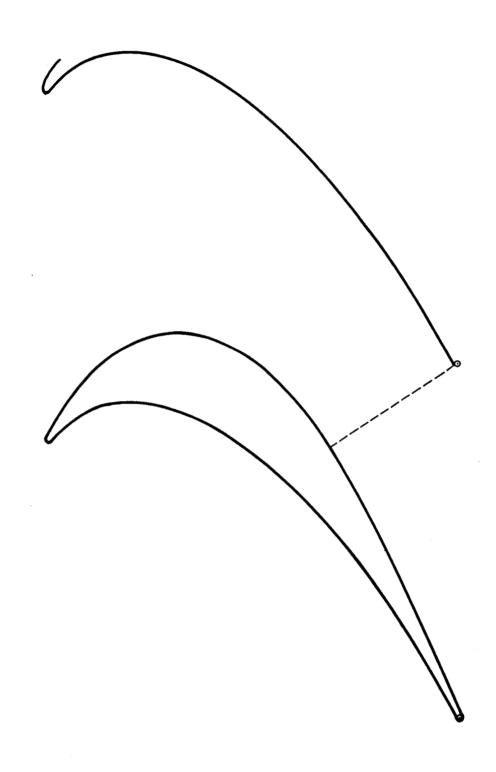


FIGURE 83 FOURTH STAGE BLADE QUARTER TIP

2.5 SCALE

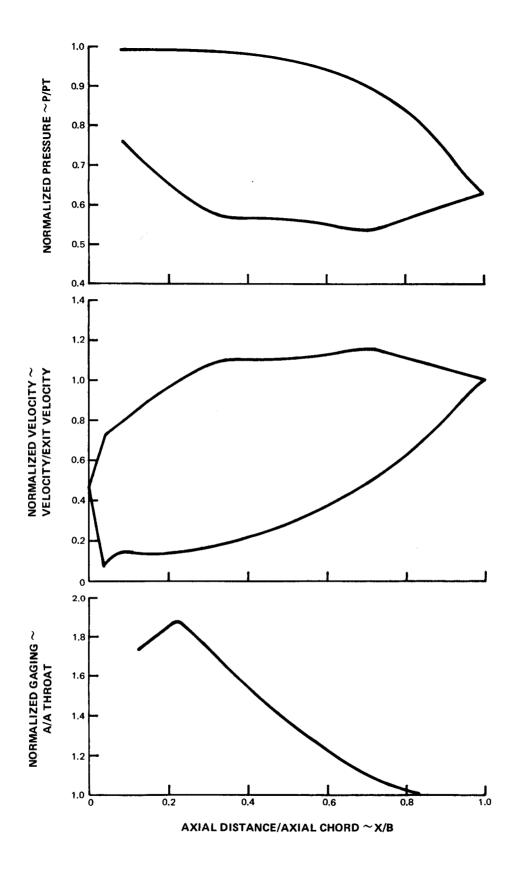


FIG. 84 FOURTH STAGE RLADE QUARTER TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

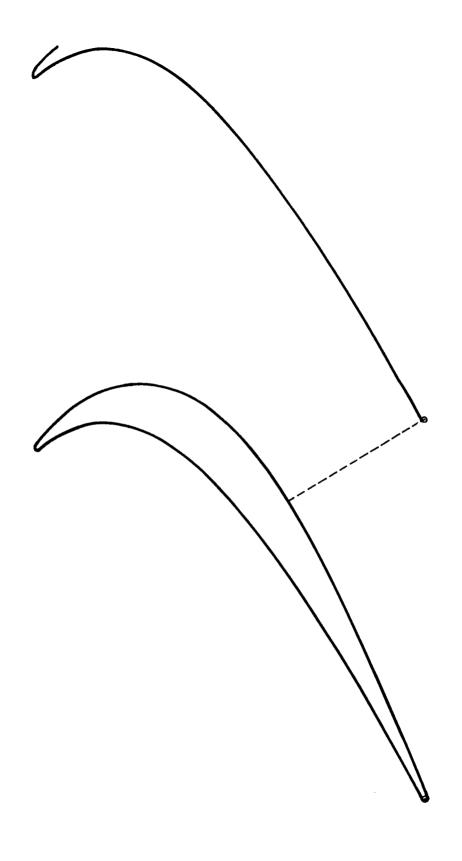


FIGURE 85 FOURTH STAGE BLADE TIP

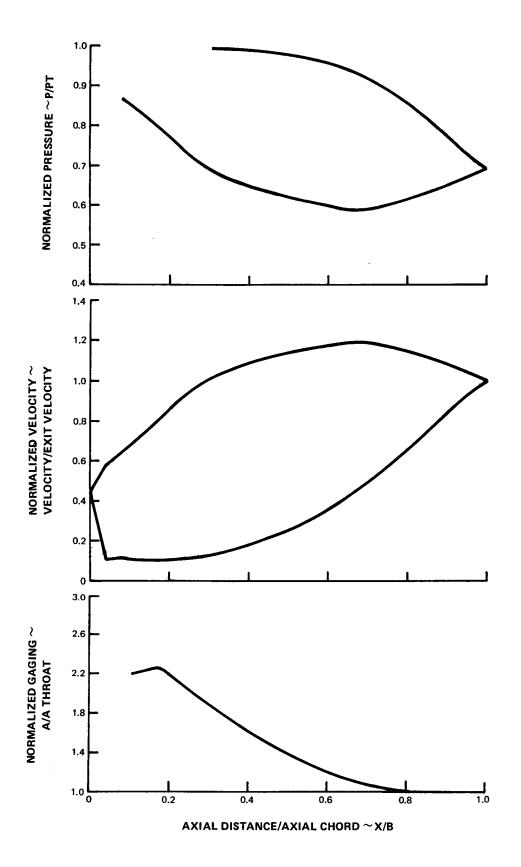


FIG. 86 FOURTH STAGE BLADE TIP NORMALIZED PRESSURE VELOCITY AND GAGING DIAGRAMS

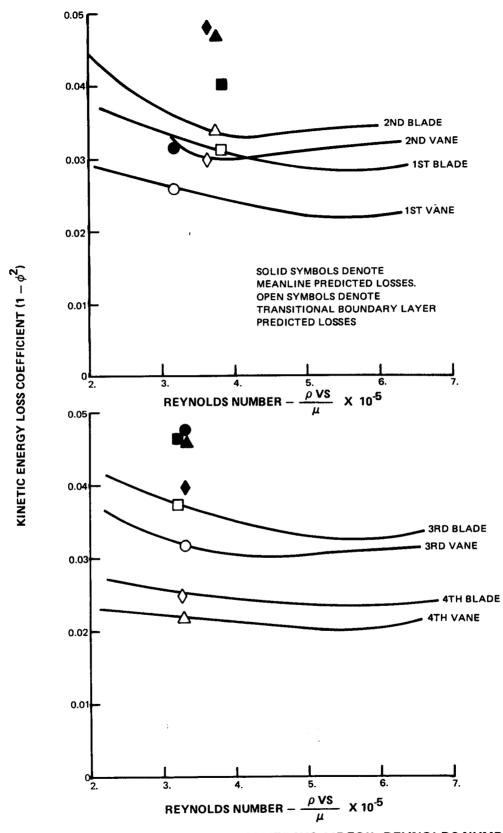


FIG. 87 AIRFOIL PROFILE LOSS VERSUS AIRFOIL REYNOLDS NUMBER

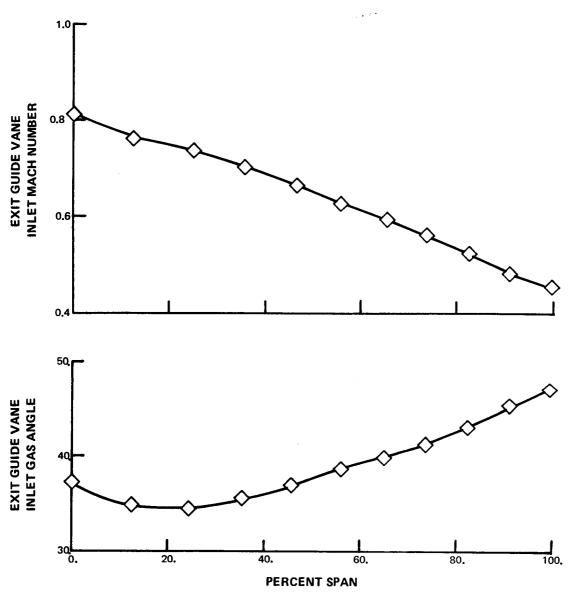


FIGURE 88 EXIT GUIDE VANE INLET CONDITIONS

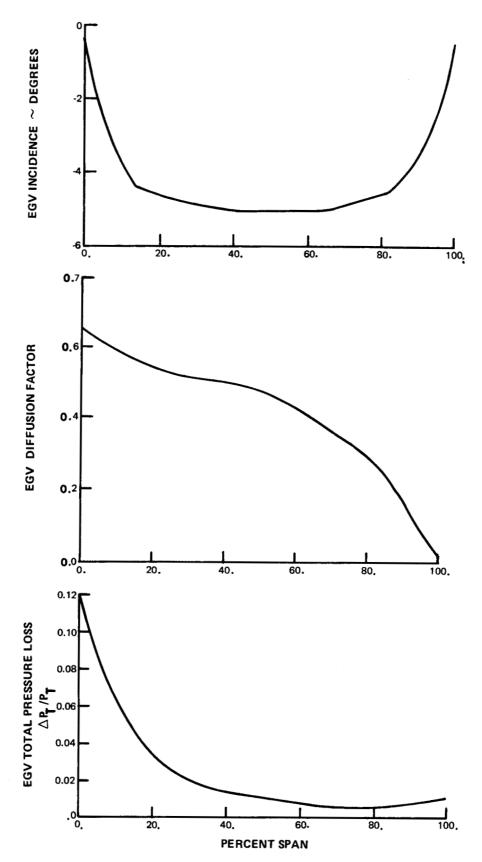


FIG. 89 EGV INCIDENCE, DIFFUSION FACTOR AND LOSS VERSUS PERCENT SPAN

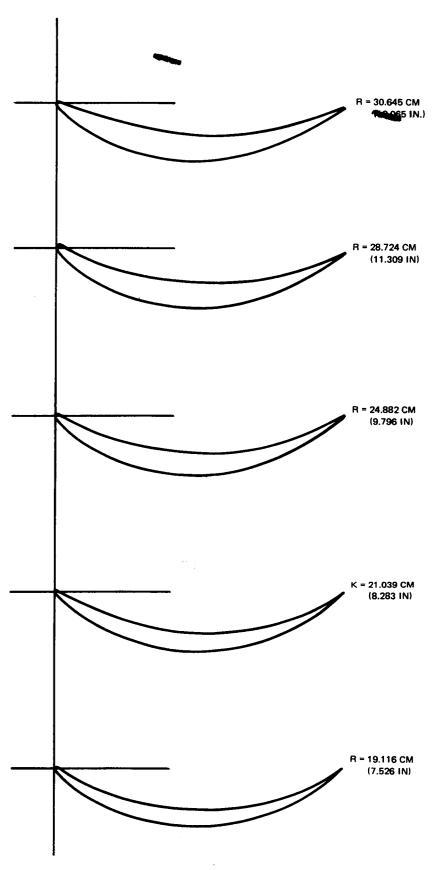
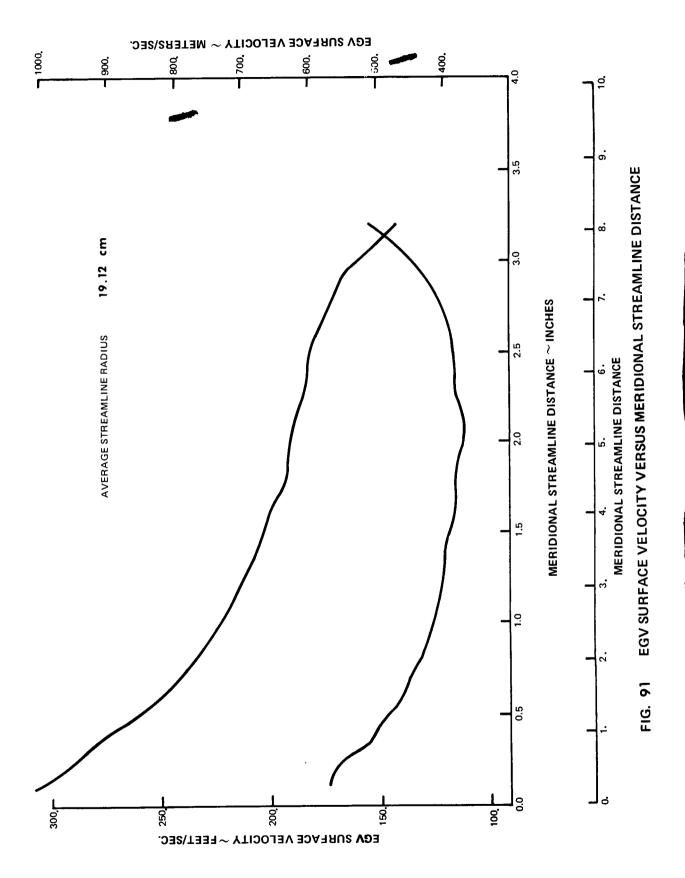
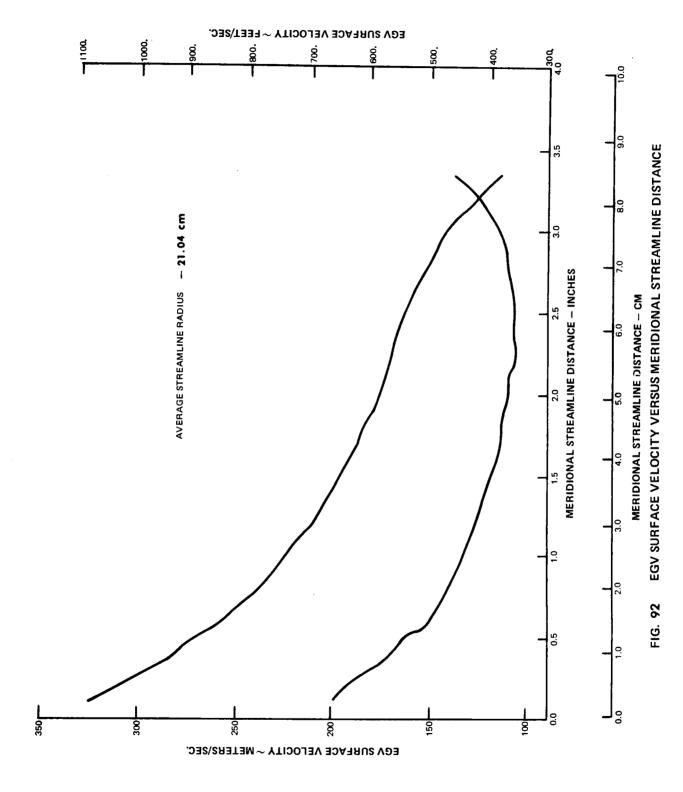


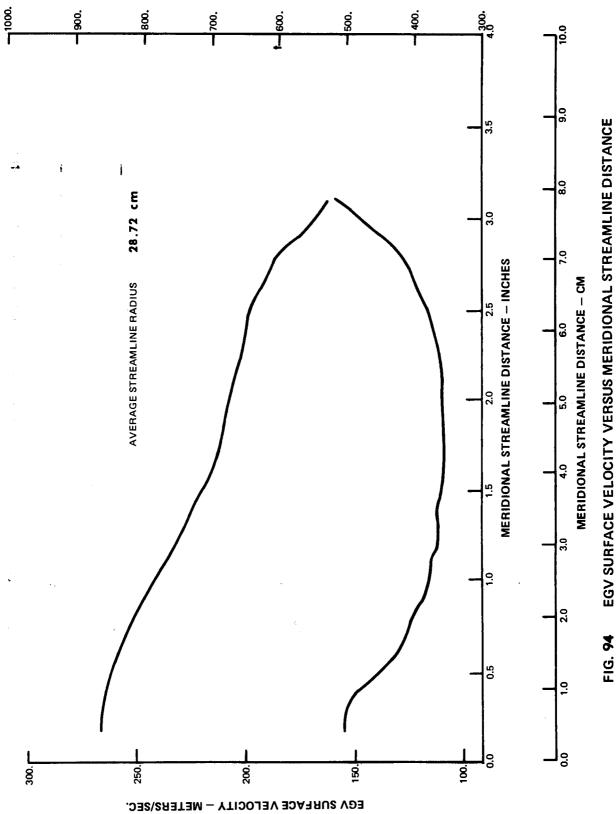
FIG. 90 TURBINE EXIT GUIDE VANE - 1.0 SCALE





EGY SURFACE VELOCITY - METERS/SEC.

EGV SURFACE VELOCITY VERSUS MERIDIONAL STREAMLINE DISTANCE FIG. 93



EGV SURFACE VELOCITY - FEET/SEC.

EGV SURFACE VELOCITY VERSUS MERIDIONAL STREAMLINE DISTANCE

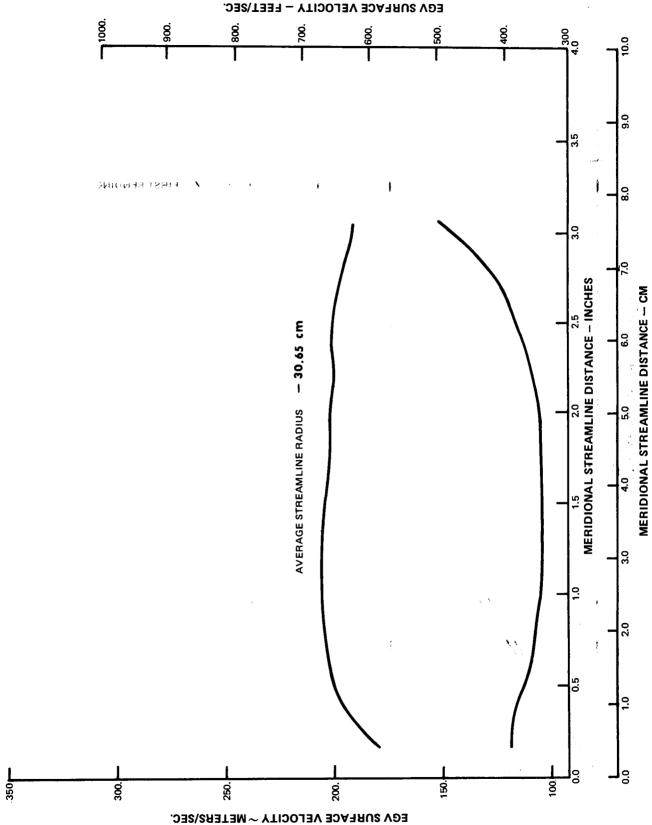
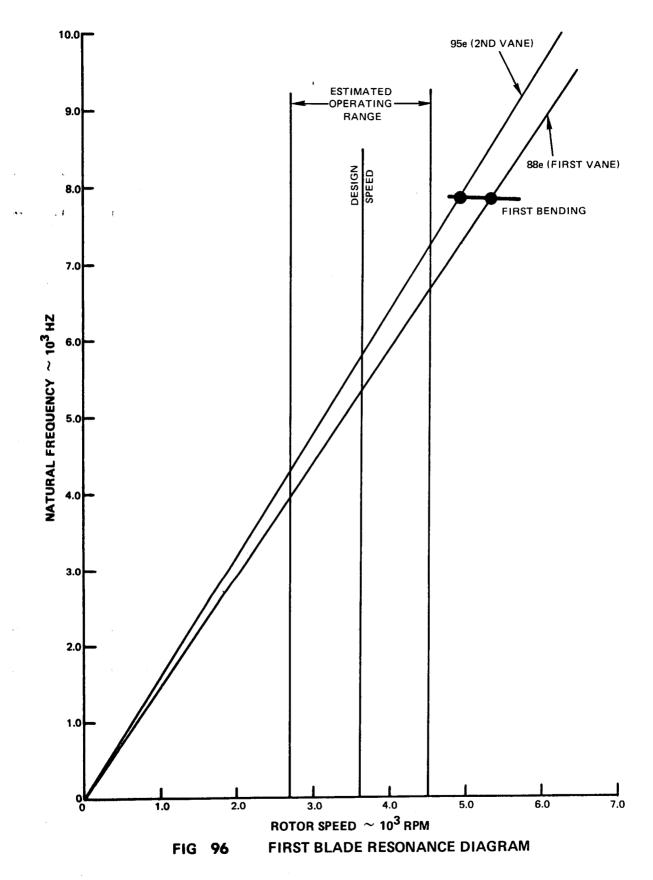


FIG. 95 EGV SURFACE VELOCITY VERSUS MERIDIONAL STREAMLINE DISTANCE



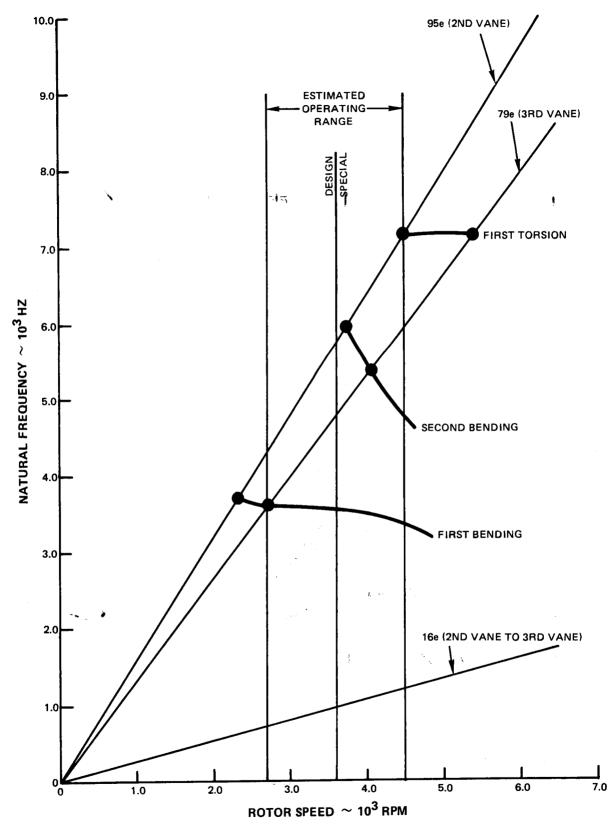


FIG. 97 SECOND BLADE RESONANCE DIAGRAM

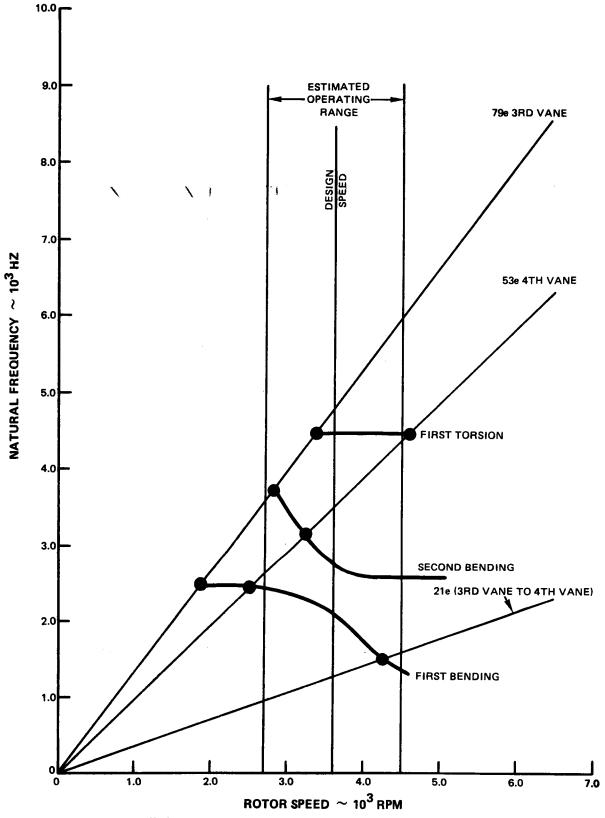


FIG. 98 THIRD BLADE RESONANCE DIAGRAM

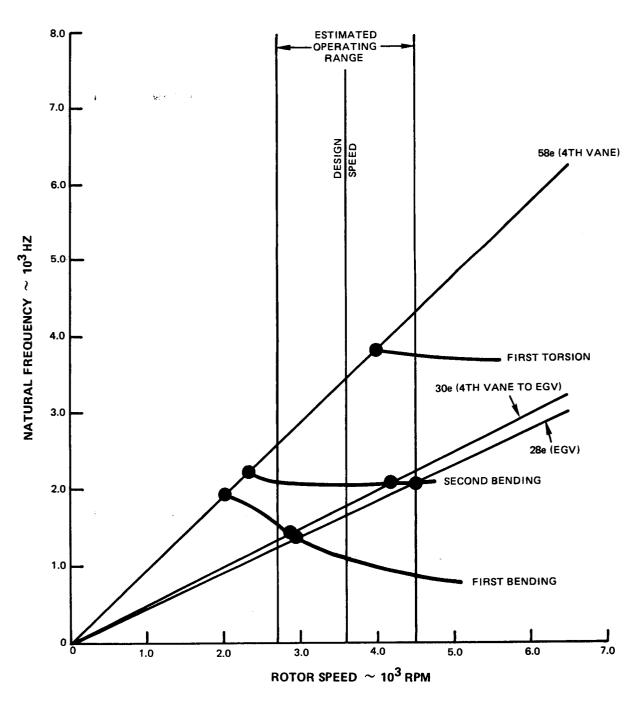
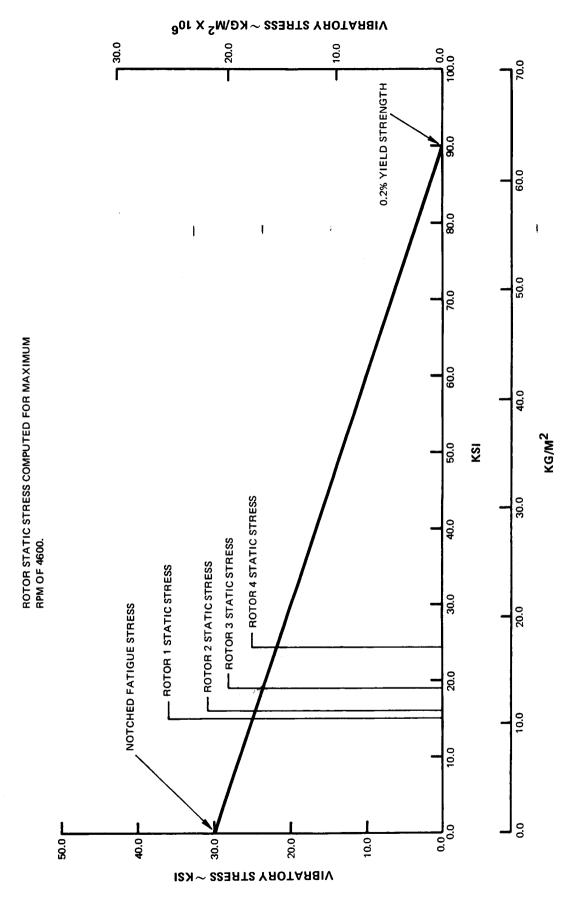


FIG. 99 FOURTH BLADE RESONANCE DIAGRAM



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ROTOR GOODMAN DIAGRAM

FIG. 100

TABLE I
TURBINE AERODYNAMICS

	1	FIRST STAGE		SE	COND STAGE	
RADIAL STATION	ROOT	MEAN	TIP	ROOT	MEAN	TIP
R ₀ (VANE INLET) CM (IN.)	20.264 (7.978)	22.240 (8.756)	24.13 (9.500)	19.632 (7.729)	20.898 (9.015)	26.025 (10.246)
, ,	19.939	22.209	24.486	19.454	23.126	26.208
R ₁ (VANE EXIT) CM (IN.)	(7.850)	(8.744)	(9.640)	(7.659)	(9.105)	(10.515)
R _{1.5} (BLADE INLET) CM	19.733	22.253	24.887	19.279	23.236	27.087
(IN.)	(7.769)	(8.761)	(9.798)	(7.590)	(9.148)	(10.664)
R2 (BLADE EXIT) CM	19.733	22.738	25.596	19.279	23:810	27.894
(IN.)	(7.769)	(8.952)	(10.077)	(7.590)	(9.374)	(10.982)
U2 WHEEL SPEED MPS	74.37	85.95	96.62	72.85	89.92	105.16
(FPS)	(244.)	(282)	(317.)	(239.)	(295.)	(345.)
∆h Joules/kg	35306.	39864.	36980.	36678.	40725.	35 515.
(BTU/Ib)	(15.18)	(17.14)	(15.90)	15.97).	(17.51)	(15.27)
VEL. RATIO	0.280	0.304	0.355	0.269	0.315	0.395
αO Deg.	90.00	90.00	90.00	32.12	28.66	31.21
α _η Deg.	31.53	25.73	23.93	26.64	21.63	20.64
$ heta_{f V}$ Deg.	58.47	64.27	66.07	121.24	129.71	128.15
$eta_{1.5}$ Deg.	37.46	31.82	29.47	31.48	28.22	32.75
β_2 Deg.	26.66	22.66	22.66	25.37	21.37	21.37
$ heta_{B}$ Deg.	115.88	125.52	127.87	123.15	130.41	125.88
ര ₂ Deg.	34.09	30.94	35.05	32.48	30.29	35.93
REACTION (PRESS.)	0.375	0.335	0.377	0.430	0.468	0.534
(Ps/Ps)V	1.457	1.434	1.375	1.285	1.249	1.213
(PS/PS)B	1.343	1.261	1.250	1.260	1.259	1.291
(C _L) _V	0.757	0.740	0.773	0.910	0.901	0.957
(CL)B	0.851	0.884	0.980	0.823	0.857	0.896
(MR)B INLET	0.600	0.573	0.456	0.698	0.584	0.375
(MR)B EXIT	0.857	0.814	0.678	0.882	0.817	0.671
(MA)VINLET	.317	.283	.300	0.674	0.589	0.431
(MA)V EXIT	.782	.782	.708	0.881	0.813	0.684
C _{X1/C} FPS X2 (MPS)	517. (476)	433.(388)	366.(327)	475.(444)	362.(350)	276.(294)
STAGE WORK						
Joules/Kg		38934.			20404	
(BTU/lb)		(16.74)			39494 . (16.09)	
(610/10)		110.171			(16.98)	

TABLE I (Cont'd)
TURBINE AERODYNAMICS

		THIRD STAGE STREAMLINE			URTH STAGE TREAMLINE	
RADIAL STATION	ROOT	MEAN	TIP	ROOT	MEAN	TIP
R ₀ (VANE INLET) CM (IN.)	19.202 (7.560)	23.944 (9.427)	28.298 (11.141)	18.519 (7.291)	25.042 (9.859)	30.698 (12.086)
R ₁ (VANE EXIT) CM (IN.)	18.849 (7.421)	24.158 (9.511)	29.108 (11.460)	18.064 (7.112)	25.194 (9.919)	31.458 (12.385)
R _{1.5} (BLADE INLET) CM	18.621 (7.331)	24.257 (9.550)	29.487 (11.609)	17.787 (7.003)	25.237 (9.936)	31.836 (12.534)
R2 (BLADE EXIT) CMFC	18.6291	724.928	30.320	17.788	26.070	32.593
U ₂ WHEEL SPEED MPS	(7.331) 70.41	(9.814) 94.18	(11.937) 114.60	(7.003) 67.06	(10.264) 98.45	(12.832) 123.14
(FPS)	(231.)	(309.)	(376.)	(220.)	(323.)	(404.)
Δ h joules/kg	34724.	38771.	33608.	33.189	38212.	34922.
(ВТU/Љ)	14.93	16.67	14.45	14.27	16.43	14.8
VEL. RATIO	0.267	0.338	0.441	0.261	0.356	0.469
$lpha_{f 0}$ Deg.	32.52	28.73	33.10	34.17	32.54	39.95
α ₁ Deg.	26.82	21.78	20.82	29.54	24.47	23.54
$ heta_{_{f V}}$ Deg.	120.67	129.49	126.09	116.29	122.99	116.51
β _{1.5} Deg.	31.86	30.05	38.80	35.55	35.60	48.74
β ₂ Deg.	26.75	22,75	22.75	29.99	25.99	25.99
θ _B Deg.	121.40	127.20	118.45	114.46	118.40	105.27
€ Deg.	34.07	33.81	42.19	37.25	38.57	47.94
REACTION	0.406	0.462	0.553	0.401	0.522	0.585
$(P_S/P_S)_V$	1.303	1.257	1.205	1.355	1.278	1.224
(PS / P S)B	1.253	1.266	1.312	1.319	1.410	1.423
(C ^L)V	0.857	0.909	0.999	0.824 #7969	0.956	1.062
(C _L) _B	0.815	0.904	0.933	0.803	0.953	0.986
(M _R) _B INLET	0.739	0.563	0.321	0.811	0.544	0.297
(MR)BEXIT	0.912	0.807	0.660	1.018	0.897	0.740
(M _A)VINLET	0.708	0.577	0.400	0.739	0.548	0.364
(MA)V EXIT	0.921	0.810	0.604	0.981	0.806	0.601
C _{X1} /C _{X2} MPS (FPS) STAGE WORK	472.(456)	344.(356)	254.(293)	517. (528)	360.(405)	271.(351)
Joules/Kg (BTU/Ib)		37445. (16.10)			36887. (15.86)	

TABLE I (Cont'd) EXIT GUIDE VANE AERODYNAMICS

RADIAL STATION	ROOT	MEAN	TIP
R _O (VANE INLET) CM.	17.787	26.167	32.822
IN.	7.003	10.302	12.922
R ₁ (VANE EXIT) CM.	20.447	26.570	31.031
IN.	8.050	10.461	12.217
α_{0}	36.16	38.41	46.18
α_1	90.0	90.0	90.0
£,	53.84	51.59	43.82
P _S /P _S	0.792ິ3 ີໍ	+0.9418 ^{€1} [©]	[™] 51.0877
(M _A) VINLET	0.8295	0.6268	0.4190
(MA) V EXIT	0.4385	0.5292	0.5440

TABLE II FIRST STAGE AIRFOILS

VANE GEOMETRY

	AVERAGE LENGTH (L) ASPECT RATIO (L/ B) =	AVERAGE LENGTH (L) = 4.206 CM ASPECT RATIO (L/B) = 1.705	HUB/TIP RATIO = 0.827 NO. OF VANES = 88).827 88 1081.1	
RADIAL STATION	ROOT	QUARTER ROOT	MEAN	QUĂRTER TIP	립
DEFINING RADIUS CM (IN.)	19.939 (7.850)	21.014 (8.297)	22.212(8.745)	23.348 (9.192)	24.486 (9.640)
AXIAL CHORD CM (IN.)	1.676 (0.660)	1.676 (0.660)	1.676 (0.660)	1.676 (0.660)	1.676 (0.660)
ACTUAL CHORD CM (IN.)	2.261(0.890)	2.393 (0.942)	2.466 (0.971)	2.543 (1.001)	2.639 (1.039)
GAP/AXIAL CHORD	0.849	0.898	0.946	10.994	1.043
α,*/α, - DEGREES	00.06/00.06	00.06/00.06	00.06/00.06	00.06/00. <u>9</u> 6	90.00/90.00
α_*/α_* - DEGREES	31.53/31.53	26.68/26.68	25.73/ 25.73	24.76/24.80	23.93/23.93
$\theta^*=(180-\alpha^*-\alpha_1^*)$ DEGREES	58.47	68.62	64.27	65.24	66.07
ರ GAGING	31.16	26.73	25.68	24.93	24.30
LED CM (IN.)	0.076 (0.030)	0.076 (0.030)	0.076 (0.030)	0.076 (0.030)	0.076(0.030)
TED CM (IN.)	0.038 (0.015)	0.038 (0.015)	0.038 (0.015)	0.038 (0.015)	0.038 (0.015)
UNCOVERED TURN DEGREES	8.068	8.097	8.449	8.826	9.232
BLADE GEOMETRY					
	AVERAGE LI	AVERAGE LENGTH (L) = 5.509 CM	HUB/TIP RATIO = 0.782	0.782	
	ASPECT RATIO (L/B)	10 (L/B) = 2.763	NO. OF BLADES = 102	102	
RADIAL STATION	ROOT	QUARTER ROOT	MEAN	QUARTER TIP	TIP
DEFINING RADIUS CM (IN.)	19.733(7.769)	21.199 (8.346)	22.664 (8.923)	24.130(9.500)	25.596 (10.077)
AXIAL CHORD CM (IN.)	1.880 (0.740)	1.880 (0.740)	1.880(0.740)	1.880 (0.740)	1.88(0.740)
ACTUAL CHORD CM (IN.)	1.887(0.743)	1.991 (0.764)	1.994(0.785)	2.068 (0.814)	2.139(0.842)
GAP/AXIAL CHORD	0.647	0.695	0.743	0.791	0.839
β_1*/β_1 DEGREES	34,44/37.44	26.8/31.8	25.60/31.60	24.0/30.0	23.47/29.47
β_2^*/β_2^* DEGREES	26.66/26.66	22.66/22.66	22.66/22.66	22.66/22.66	22.66/22.66
$\theta^*_{=}(18\tilde{0}-B_1^*-B_2^*)$ DEGREES	118.90	130.5	131.74	133.34	133.87
eta_2 gaging - Degrees	26.90	23.00	22.73	22.40	22.05
LED CM (IN.)	0.076 (0.130)	0.076 (0.030)	0.076 (0.030)	0.076(0.030)	0.076(0.030)
TED CM (IN.)	0.038(0.015)	0.038 (0.015)	0.038(0.015)	0.038(0.015)	0.038 (0.015)
UNCOVERED TURN - DEGREES	14.735	13.07	12.516	12.20	12.852

TABLE II (Cont'd) SECOND STAGE AIRFOILS

VANE GEOMETRY

\$4 A4	QUARTER TIP TIP 24.895 (9.801) 26.708(10.515) 1.981(0.780) 1.981(0.780) 2.273 (0.895) 2.383 (0.938) 0.831 0.892 24.4/30.4 25.60/31.60 21.1/21.1 20.64/20.64 134.5 133.76 21.1 20.48 0.076 (0.030) 0.038 (0.015) 11.56 11.674	OUARTER TIP 27.894(10.982) 2.337 (0.920) 2.337 (0.920) 2.337 (0.920) 2.337 (0.920) 2.0.805 24.0/30.0 26.73/32.73 21.37/21.37 21.37/21.37 134.63 131.90 21.15 20.85 0.076 (0.030) 0.038 (0.015)
HUB/TIP RATIO = .741 NO. OF VANES = 95	MEAN 23.081 (0.087) 1.981 (0.780) 2.149 (0.846) 0.771 22.80/ 28.80 21.63/ 21.63 135.57 21.64 0.076 (0.030) 0.038 (0.015)	HUB/TIP RATIO = .701 NO. OF BLADES = 86 23.586 (9.286) 2.337 (0.920) 2.428 (0.956) 0.737 22.40/28.40 21.37/21.37 136.23 21.40 0.076 (0.030) 0.038 (0.015)
AVERAGE LENGTH = 6.825 CM. ASPECT RATIO (L/ B) = 3.176	21.267 (8.373) 1.981 (0.780) 2.019 (0.795) 0.710 22.5/27.5 22.2/22.2 135.3 22.2 0.076 (0.030) 0.038 (0.015	AVERAGE LENGTH (L) = 82/2 CM ASPECT RATIO (L/B) = 3.382 L590)
AVERAGE L	ROOT 19.454 (7.659) 1.981 (0.780) 2.002 (0.788) 0.649 30.30/33.30 26.64/26.64 123.06 26.64 0.076 (0.030) 0.038 (0.015)	AVERAC ASPECT ASPECT 19.279 (7.590) 2.350 (0.925) 0.603 26.48/31.48 25.37/25.37 128.15 25.57 0.076 (0.030) 0.038 (0.015)
	RADIAL STATION DEFINING RADIUS CM (IN.) AXIAL CHORD CM (IN.) ACTUAL CHORD CM (IN.) GAP/AXIAL CHORD $\alpha_0^{*/\alpha_0} - \text{DEGREES}$ $\alpha_1^{*/\alpha_1} - \text{DEGREES}$ $\alpha_1^{*/\alpha_1} - \text{DEGREES}$ $\alpha_1^{*} = (180 - \alpha_0^* - \alpha_1^*) - \text{DEGREES}$ $\alpha_1^{*} = (180 - \alpha_0^* - \alpha_1^*) - \text{DEGREES}$ $\alpha_1^{*} = (180 - \alpha_0^* - \alpha_1^*) - \text{DEGREES}$ $\alpha_1^{*} = (180 - \alpha_0^* - \alpha_1^*) - \text{DEGREES}$ LED $\alpha_1^{*} = (180 - \alpha_0^* - \alpha_1^*) - \text{DEGREES}$ LED $\alpha_1^{*} = (180 - \alpha_0^* - \alpha_1^*) - \text{DEGREES}$ UNCOVERED TURN $\sim \text{DEGREES}$	BLADE GEOMETRY RADIAL STATION DEFINING RADIUS CM(IN.) AXIAL CHORD CM (IN.) ACTUAL CHORD CM (IN.) GAP/AXIAL CHORD $\beta_1*/\beta_1* \text{ DEGREES}$ $\beta_2*/\beta_2 \text{ DEGREES}$ $\beta_2*/\beta_2 \text{ DEGREES}$ $\beta_2*/\beta_3 \text{ DEGREES}$ $\beta_2^2*/\beta_2 \text{ DEGREES}$ $\beta_2^2*/\beta_2 \text{ DEGREES}$ $\beta_2^2*/\beta_2 \text{ DEGREES}$ $\beta_1*/\beta_1* \text{ DEGREES}$ $\beta_2^2*/\beta_2 \text{ DEGREES}$ $\beta_2^2*/\beta_2 \text{ DEGREES}$ $\beta_1*/\beta_1* \text{ DEGREES}$ $\beta_2^2*/\beta_2 \text{ DEGREES}$ $\beta_1*/\beta_1* \text{ DEGREES}$ $\beta_2^2*/\beta_2 \text{ DEGREES}$ $\beta_1*/\beta_1* \text{ DEGREES}$ $\beta_2^2*/\beta_2 \text{ DEGREES}$ $\beta_2^2\text{ GAGING}$ LED $CM (IN.)$ TED $CM (IN.)$

TABLE II (Cont'd) THIRD STAGE AIRFOILS

HUB/TIP RATIO = 0.663 NO. OF VANES = 79

AVERAGE LENGTH (L) = 9.678 CM ASPECT RATIO (L/B) = 3.605

VANE GEOMETRY

RADIAI STATION	ROOT	OUARTER ROOT	MEAN	OUARTER TIP	di
DEFINING RADIUS CM(IN.)	18.847 (7.420)	21.412 (8.430)	23.978 (9.440)	26.543 (10.450)	29.108 (11.460)
AXIAL CHORD CM (IN.)	2.464(0.970)	2.464 (0.970)	2.464 (0.970)	2,464 (0.970)	2.464 (0.970)
ACTUAL CHORD CM (IN.)	2.499(0.984)	2.543 (1.001)	2.685 (1.057)	2.893 (1.139)	3.170(1.248)
GAP/AXIAL CHORD	0.608	0.691	0.774	ੁਂ 0.857	0.940
αη*/αη — DEGREES	28.90/33.90	22.70/27.22	22.80/28.80	25.13/31.13	27.80/33.80
α,*/α, – DEGREES	26.82/26.82	22.38/22.38	21.81/21.81	21.24/21.34	20.82/20.82
θ^* =(180 - α_0^* - α_1^*) DEGREES	124.28	134.92	135.39	133.53	131.38
$lpha_{\sf q}$ GAGINĞ — DEGREES	27.02	22.46	21.78	21.24	20.60
LED CM (IN.)	0.076 (0.030)	0.076 (0.030)	0.076(0.030)	0.076 (0.030)	0.076(0.030)
TED CM (IN.)	0.038 (0.015)	0.038 (0.015)	0.038(0.015)	0.038 (0.015)	0.038 (0.015)
UNCOVERED TURN - DEGREES	12.271	11.73	10.129	10.72	11.165
BLADE GEOMETRY				2 %	
	AVERAC	AVERAGE LENGTH (L) = 11.282 CM	CM HUB/TIP RATIO = 0.623	.623	
	ASPECT	ASPECT RATIO (L/B) = 3.702	NO. OF BLADES =	72	
RADIAL STATION	ROOT	QUARTER ROOT	MEAN	QUARTER TIP	TIP
DEFINING RADIUS CM(IN.)	18.621(7.331)	21.547 (8.483)	24.600 (9.685)	27.394 (10.785)	30.320(11.937)
AXIAL CHORD CM (IN.)	2.870(1.130)	2.870 (1.130)	2.870 (1.130)	2:870 (1.130)	2.870 1.130)
ACTUAL CHORD CM (IN.)	2.865(1.128)	2.916 (1.148)	3.048 (1.200)	3.279 (1.291)	3.617 (1.424)
GAP/AXIAL CHORD	0.566	0.655	0.748	0.833	0.922
β_1*/β_1 – DEGREES	28.84/31.84	23.5/28.50	24.05/30.05	27.35/33.35	32.78/38.78
β_2^*/β_2 – DEGREES	26.75/26.75	22.9/22.9	22.75/22.75	22.75/22.75	22.75/ 22.75
$\theta = (180-B_1^* - B_2^*)$ DEGREES	124.41	133.6	133.20	129.9	124.47
B ₂ GAGING – DEGREES	26.99	23.0	22.60	22.40	21.93
LED CM(IN.)	0.076(0.030)	0.076 (0.030)	0.076 (0.030)	0.076 (0.030)	0.076(0.030)
TED CM (IN.)	0.038 (0.015)	0.038 (0.015)	0.038 (0.015)	0.038(0.015)	0.038(0.015)
UNCOVERED TURN – DEGREES	10.951	11.13	10.593	11.18	10.437

TABLE II (Cont'd) FOURTH STAGE AIRFOILS

VANE GEOMETRY

	AVERAG	AVERAGE LENGTH $(L) = 12.786 \text{ CM}$	HUB/TIP RATIO = .589	.589	
	ASPECT	ASPECT RATIO (L/B) ∓ 3.431	NO. OF VANES =	58	
RADIAL STATION	ROOT	QUARTER ROOT	MEAN	QUARTER TIP	TIP
DEFINING RADIUS CM (IN.)	18.064 (7.112)	21.412 (8.430)	24.762 (9.749)	28.110 (11.067)	31.458 (12.385)
AXIAL CHORD CM (IN.)	3.556 (1,400)	3.556 (1.400)	3.556 (1.400)	3.556 (1.400)	3.556 (1.400)
۵	3.534 (1.407)	3.589 (1.413)	3.726 (1.467)	3.975 (1.565)	4.392 (1.729)
GAP/AXIAL CHORD CM (IN.)	0.550	0.652	0.754	0.856	0.958
α,*/α, – DEGREES	34.30/37.30	24.43/29.92	26.30/32.30	30.1/36.0	35.30/ 41.30
α,*/α, – DEGREES	29.54/29.54	25.1/ 25.1	24.52/24.52	24.0/24.02	22.97/ 22.97
θ^* =(180 · α_n^* · α_1^*) DEGREES	116.16	130.47	129.18	125.88	121.73
lpha, GAGINĞ DEGREES	29.87	25.12	24.30	23.66	22.97
LED CM(IN.)	0.076(0.030)	0.076 (0.030)	0.076(0.030)	0.076 (0.030)	0.076 (0.030)
TED CM(IN.)	0.038 (0.015)	0.038 (0.015)	0.038 (0.015)	0.038 (0.015)	0.038 (0.015)
UNCOVERED TURN DEGREES	12.010	10.58	10.027	11.23	12.27
BLADE GEOMETRY					
	AVERAGI	AVERAGE LENGTH (L) = 14.427 CM	HUB/TIP RATIO = .552	.552	
	ASPECT F	ASPECT RATIO (L/B) = 3.111	NO. OF BLADES =	50 ,	
RADIAL STATION	ROOT	QUARTER ROOT	MEAN	OUARTER TIP	TIP
DEFINING RADIUS CM(IN.)	17.788 (7.003)	21.488 (8.460)	25.190 (9.9175)	28.890 (11.374)	32.593 (12.832)
AXIAL CHORD CM(IN.)	4.2672 (1.680)	4.267 (1.680)	4.267 (1.680)	4.267 (1.680)	4.267 (1.680)
ACTUAL CHORD CM (IN.)	4.313 (1.698)	4.361 (1.717)	4.638 (1.826)	5.108 (2.011)	5.687 (2.239)
GAP/AXIAL CHORD	0.524	0.633	0.742	0.851	0.960
β_1^*/β_1 – DEGREE Sq	32.50/35.50	27.6/32.6	29.51,34.80	34.38/40.4	46.00/ 52.00
β_2^*/β_2 – DEGREES	29.99/29.99	26.1/26.1	25.9/ 25.95	25.99/25.99	25.99/25.99
<i>θ*</i> Ξ(180-Β ₁ * - Β ₂ *) DEGREES	117.51	126.3	124.54	- 119.63	108.01
B, GAGING – ĎEGREES	30.69	26.58	26.00	25.70	24.84
LED CM (IN.)	0.076 (0.030)	0.076 (0.030)	0.076(0.030)	0.076 (0.030)	0.036(0.030)
TED CM (IN.)	0.038 (0.015)	0.038 (0.015)	0.038(0.015)	0.038 (0.015)	0.038 (0.015)
UNCOVERED TURN – DEGREES	8.935	10.58	9.872	8.7	8.786

TABLE 111
FIRST STAGE VANE
NON-DIMENSIONAL AIRFOIL COORDINATES

	1000	1	QUARTER	TER	uw	MEAN	QUARTER	TER	P	
_ l S	YS/8*	YP/B*	ROOT YS/B	OT YP/B	YS/B	YP/B	YS/B	P YP/B	YS/B	YP/B
.8983 0.9227	2 23	.8983	1.0394	1.01 4 2 0.9877	1.0722 1.0980	1.0449	1.1349	1.1349	1.2074 1.2330	1.207 4 1.1798
0.9223	 g	8579	1.0415	0.9780	1.1008	1.0339	1.1626	1.0959	1.2341	1.1668
0.9172	22	.8383	1.0399	0.9638	1.0993	1.0188	1.1597	1.0795	1.2295	1.1477
0.9069	66	.8136	1.0341	0.9448	1.0929	1999.1	1.1511	1.057?	1.2183	1.1217
0.8911	=	.7848	1.0230	0.9209	1.0806	.9744	1.1361	1.0302	1.2000	1.0885
0.8695	32	.7521	1.0056	0.8917	1.0615	.9443	1.1139	.9963	1.1737	1.0479
0.8419	61	.7158	0.9810	0.8571	1.0348	.9082	1.0838	.9555	1.1388	1.0002
0.8082	32	.6762	0.9484	0.8171	9666.0	.8655	1.0452	7206.	1.0952	0.9460
0.7684	34	.6335	0.9070	0.7714	0.9556	.8159	0.9977	.8534	1.0425	0.8862
0.7224	24	.5876	0.8567	0.7203	0.9025	.7600	0.9414	.7930	0.9812	0.8211
0.6706	90	.5385	0.7978	0.6639	0.8406	0869	0.8764	.7272	0.9114	0.7513
0.6131	75	.4865	0.7309	0.6022	0.7705	.6309	0.8033	.6565	0.8340	0.6771
.5502	22	.4316	0.6569	0.5355	0.6929	.5589	0.7226	.5814	0.7495	0.5988
.4824	54	.3736	0.5768	0.4641	0.6088	.4829	0.6352	.5020	0.6588	0.5167
.4101		.3126	0.4914	0.3884	0.5190	.4029	0.5417	.4189	0.5622	0.4308
.3341	=	.2486	0.4013	0.3085	0.4240	.3192	0.4427	.3320	0.4602	0.3414
.2548	<u> </u>	.1816	0.3070	0.2449	0.3244	.2324	0.3388	.2418	0.3531	0.2486
.1728	8:	.1114	0.2089	0.1378	0.2207	.1423	0.2305	.1482	0.2410	0.1524
.0888	88	.0380	0.1073	0.0476	0.1132	.0492	0.1182	.0514	0.1240	0.0530
0.0000	0	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.000.0

X/B = NON DIMENSIONAL AXIAL COORDINATE USING THE AXIAL CHORD
YS/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE SUCTION SURFACE USING THE AXIAL CHORD
YP/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE PRESSURE SURFACE USING THE AXIAL CHORD * X/B

TABLE III (CONTINUED)
FIRST STAGE BLADE
NON-DIMENSIONAL AIRFOIL COORDINATES

N. (1.19

-		+ (QUARTER	TER	1980		OUARTER	TER		-
	HQ.	ROOI	ROOT)T	N.C. III	2.	TIP		dit :	a.
	YS/8*	¥8/4X	YS/B	4P/8	YS/B	YP/B	YS/B	YP/B	YS/B	YP/B
	0.1939	0.1939	0.3248	0.3248	0.4072	0.4072	0.5042	0.5042	0.5818	0.5818
	0.2808	0.1975	0.4374	0.3329	0.5234	0.4166	0.6261	0.5151	0.7053	0.5935
	0.3513	0.2439	0.5239	0.3847	0.6099	0.4738	0.7132	0.5765	0.7919	0.6585
	0.4119	0.2773	0.5933	0.4161	0.6773	0.5087	0.7782	0.6129	0.8551	0.6970
	0.4634	0.3018	0.6484	0.4356	0.7291	0.5300	0.8260	0.6336	0.9001	0.7179
	0.5064	0.3194	0.6912	0.4468	0.7677	0.5414	0.8594	0.6429	0.9299	0.7258
	0.5411	0.3311	0.7227	0.4515	0.7946	0.5450	0.8802	0.6431	0.9466	0.7232
	0.5678	0.3378	0.7438	0.4507	0.8107	0.5419	0.8897	0.6355	0.9515	0.7116
	0.5864	0.3400	0.7550	0.4450	0.8166	0.5330	0.8885	0.6212	0.9453	0.6923
	0.5970	0.3378	0.7565	0.4348	0.8126	0.5186	0.8771	9009:0	0.9287	0.6658
	0.5993	0.3315	0.7483	0.4203	0.7988	0.4991	0.8557	0.5742	0.9018	0.6328
	0.5927	0.3212	0.7303	0.4017	0.7750	0.4746	0.8242	0.5422	0.8647	0.5936
	0.5768	0.3067	0.7021	0.3789	0.7408	0.4451	0.7822	0.5046	0.8171	0.5484
	0.5507	0.2879	0.6629	0.3517	0.6955	0.4105	0.7292	0.4614	0.7587	0.4972
	0.5129	0.2645	0.6116	0.3199	0.6379	0.3705	0.6642	0.4124	0.6886	0.4400
	0.4617	0.2360	0.5467	0.2830	0.5664	0.3246	0.5857	0.3572	0.6057	0.3768
	0.3941	0.2017	0.4656	0.2400	0.4786	0.2720	0.4922	0.2953	0.5088	0.3073
	0.3098	0.1602	0.3672	0.1898	0.3745	0.2116	0.3840	0.2257	0.3982	0.2311
	0.2134	0.1096	0.2543	0.1298	0.2578	0.1413	0.2641	0.1473	0.2753	0.1477
	0.1098	0.0456	0.1313	0.0548	0.1325	0.0573	0.1358	0.0577	0.1424	0.0563
	0.0000	0.0000	0.000	0.000	0.0000	0.000	0.0000	0.0000	0.0000	0.0000

• X/B = NON DIMENSIONAL AXIAL COORDINATE USING THE AXIAL CHORD
YS/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE SUCTION SURFACE USING THE AXIAL CHORD
YP/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE PRESSURE SURFACE USING THE AXIAL CHORD

TABLE III (CONTINUED)
SECOND STAGE VANE
NON-DIMENSIONAL AIRFOIL COORDINATES

1.23 4

							-															
ТІР	YP/B	0.6021	0.7111	0.7617	0.7877	0.7983	0.7975	0.7876	0.7700	0.7455	0.7148	0.6783	0.6361	0.5883	0.5348	0.4754	0.4098	0.3374	0.2573	0.1680	0.0673	0.0000
1	YS/B	0.6021	0.8147	0.8944	0.9518	0.9921	1.0185	1.0316	1.0338	1.0253	1.0066	77760	0.9384	0.8882	0.8261	0.7509	0.6601	0.5522	0.4286	0.2933	0.1500	0.0000
QUARTER TIP	YP/B	0.4662	0.6148	0.6659	0.6923	0.7042	0.7056	0.6987	0.6850	0.6651	96290	0.6089	0.5730	0.5321	0.4860	0.4344	0.3769	0.3128	0.2409	0.1595	0.0652	0.000.0
QUAF T	YS/B	0.4662	0.7244	0.8090	0.8705	0.9145	0.9440	0.9610	0.9666	0.9618	0.9467	0.9217	0.8864	0.8406	0.7833	0.7134	0.6289	0.5276	0.4105	0.2814	0.1441	0.0000
AN	YP/B	0.2816	0.4807	0.5386	0.5722	0.5914	0.6003	0.6011	0.5950	0.5829	0.5651	0.5421	0.5141	0.4809	0.4425	0.3986	0.3487	0.2920	0.2272	0.1523	0.0633	0.0000
MEAN	YS/B	0.2816	0.6012	0.6977	0.7697	0.8230	0.8609	0.8854	0.8979	0.8991	0.8897	0.8697	0.8392	0.7977	0.7447	0.6792	0.5996	0.5040	0.3927	0.2694	0.1381	0.0000
QUARTER ROOT	YP/B	0.2816	0.2980	0.3669	0.4131	0.4450	0.4664	0.4793	0.4850	0.4842	0.4775	0.4652	0.4473	0.4240	0.3950	0.3600	0.3185	0.2697	0.2121	0.1437	0.0603	0.0000
QUAF RO	YS/B	0.2816	0.4224	0.5294	0.6136	0.6796	0.7300	0.7668	0.7911	9:08:0	0.8049	0.7951	0.7743	0.7421	0869.0	0.6412	0.5702	0.4833	0.3797	0.2624	0.1352	0.000
воот	∗8/dA	0.2234	0.2330	0.2868	0.3250	0.3524	0.3715	0.3837	0.3900	0.3908	0.3867	0.3777	0.3641	0.3457	0.3225	0.2942	0.2603	0.2202	0.1728	0.1165	0.0480	0.0000
RO	YS/8*	0.2234	0.3219	0.4020	0.4694	0.5253	0.5703	0.6050	0.6298	0.6448	0.6503	0.6460	0.6320	0.6077	0.5729	0.5268	0.4685	0.3969	0.3118	0.2157	0.1116	0.0000
*8/X		0.0	8.	.10	.15	.20	.25	<i></i> е.	.35	.40	.45	.50	.55	8	.65	07.	.75	89.	.85	06:	96:	1.0

* X/B = NON DIMENSIONAL AXIAL COORDINATE USING THE AXIAL CHORD

YS/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE SUCTION SURFACE USING THE AXIAL CHORD

YP/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE PRESSURE SURFACE USING THE AXIAL CHORD

TABLE III (CONTINUED)
SECOND STAGE BLADE
NON-DIMENSIONAL AIRFOIL COORDINATES

• X/B = NON DIMENSIONAL AXIAL COORDINATE USING THE AXIAL CHORD
YS/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE SUCTION SURFACE USING THE AXIAL CHORD
YP/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE PRESSURE SURFACE USING THE AXIAL CHORD

TABLE III (CONTINUED)
THIRD STAGE VANE
NON-DIMENSIONAL AIRFOIL COORDINATES

								,		
*/×	RO	ROOT	ROOT	TER JT	ME.	WEAN	QUARTER TIP	TER P	TIP	a
2	YS/B*	YP/B*	YS/B	YP/B	YS/B	YP/B	YS/B	YP/B	YS/B	YP/B
0.0	0.2344	0.2344	0.3126	0.3126	0.4689	0.4689	0.6447	0.6447	000	
89.	0.3315	0.2590	0.4471	0.3426	0.5992	0.4978	0.7603	0.6688	0.9314	0.8495
.10	0.4108	0.3212	0.5504	0.4070	0.6961	0.5568	0.8436	0.7155	1.0005	0.8833
.15	0.4772	0.3678	0.6317	0.4498	0.7703	0.5928	0.9052	0.7393	1.0488	0.8940
.20	0.5318	0.4024	0.6951	0.4787	0.8262	0.6143	0.9496	0.7488	1.0807	0.8908
.25	0.5753	0.4272	0.7431	0.4974	0.8666	0.6250	0.9793	0.7478	1.0989	0.8776
8	0.6082	0.4437	0.7775	0.5076	0.8933	0.6271	0.9961	0.7386	1.1050	0.8565
35.	0.6310	0.4526	0.7995	0.5107	0.9076	0.6218	1.0011	0.7224	1.1000	0.8286
04.	0.6440	0.4547	0.8097	0.5075	0.9101	0.6100	0.9949	0.7000	1.0844	0.7950
54.	0.6472	0.4504	0.8088	0.4983	0.9013	0.5921	0.9780	0.6718	1.0587	0.7559
.50	0.6408	0.4399	0.7969	0.4836	0.8814	0.5686	0.9504	0.6383	1.0229	0.7119
.55	0.6247	0.4235	0.7740	0.4634	0.8503	0.5396	0.9119	0.5997	0.9768	0.6630
99.	0.5987	0.4011	0.7400	0.4378	0.8078	0.5051	0.8623	0.5558	0.9197	0.6094
.65	0.5624	0.3727	0.6945	0.4067	0.7532	0.4650	0.8007	0.5068	0.8510	0.5509
07.	0.5156	0.3382	0.6368	0.3697	0.6855	0.4190	0.7260	0.4523	0.7691	0.4874
.75	0.4575	0.2973	0.5659	0.3264	0.6034	0.3666	0.6364	0.3919	0.6720	0.4186
8.	0.3875	0.2494	0.4802	0.2760	0.5051	0.3072	0.5307	0.3251	0.5591	0.3439
85	0.3051	0.1939	0.3789	0.2173	0.3919	0.2397	0.4108	0.2508	0.4322	0.2626
06:	0.2118	0.1300	0.2633	0.1486	0.2678	0.1622	0.2804	0.1676	0.2949	0.1735
.95	0.1098	0.0560	0.1364	0.0663	0.1367	0.0716	0.1431	0.0729	0.1504	0.0744
1.0	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.000.0

X/B = NON DIMENSIONAL AXIAL COORDINATE USING THE AXIAL CHORD
 YS/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE SUCTION SURFACE USING THE AXIAL CHORD
 YP/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE PRESSURE SURFACE USING THE AXIAL CHORD

TABLE (II (CONTINUED)
THIRD STAGE BLADE
NON-DIMENSIONAL AIRFOIL COORDINATES

X/B = NON DIMENSIONAL AXIAL COORDINATE USING THE AXIAL CHORD
YS/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE SUCTION SURFACE USING THE AXIAL CHORD
YP/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE PRESSURE SURFACE USING THE AXIAL CHORD * X/B

TABLE III (CONTINUED)
FOURTH STAGE VANE
NON-DIMENSIONAL AIRFOIL COORDINATES

×/8*	RO	ROOT	QUARTER ROOT	ater ot	ME	MEAN	QUARTER TIP	TER P	11	TIP
	YS/B*	YP/B*	A/S/B	YP/B	YS/B	YP/B	YS/B	YP/B	YS/B	YP/B
0.0	0.1574	0.1574	0.1968	0.1968	0.3444	0.3444	0.5215	0.5215	0.5215	0.5215
86	0.2370	0.1885	0.3121	0.2427	0.4613	0.3829	0.6250	0.5529	0.8238	0.7609
01.	0.3052	0.2375	0.4059	0.3048	0.5535	0.4381	0.7049	0.6002	0.8868	0.7956
.15	0.3649	0.2752	0.4835	0.3487	0.6276	0.4772	0.7678	0.6331	0.9338	0.8167
8	0.4163	0.3042	0.5473	0.3805	0.6864	0.5044	0.8162	0.6543	0.9675	0.8263
.25	0.4594	0.3258	0.5986	0.4029	0.7318	0.5220	0.8518	0.6656	0.9892	0.8260
<u>چ</u>	0.4942	0.3410	0.6384	0.4176	0.7651	0.5316	0.8757	0.6681	4.0002	0.8169
.35	0.5207	0.3506	0.6674	0.4258	0.7869	0.5340	0.8887	0.6627	1.0010	0.7996
6	0.5387	0.3550	0989'0	0.4280	0.7978	0.5300	0.8909	0.6500	0.9919	0.7750
.45	0.5480	0.3543	0.6942	0.4246	0.7979	0.5198	0.8827	0.6304	0.9730	0.7433
.50	0.5485	0.3489	0.6922	0.4160	0.7871	0.5038	0.8639	0.6042	0.9440	0.7050
529	0.5399	0.3387	0.6797	0.4022	0.7653	0.4821	0.8340	0.5717	0.9047	0.6603
99.	0.5219	0.3237	0.6561	0.3833	0.7318	0.4547	0.7924	0.5330	0.8540	0.6094
39	0.4940	0.3039	0.6208	0.3591	0.6857	0.4215	0.7380	0.4879	0.7909	0.5523
۷.	0.4557	0.2790	0.5726	0.3294	0.6256	0.3822	0.6692	0.4366	0.7132	0.4892
.75	0.4065	0.2485	0.5098	0.2936	0.5494	0.3366	0.5836	0.3787	0.6191	0.4199
œ.	0.3454	0.2121	0.4303	0.2511	0.4558	0.2840	0.4818	0.3140	0.5102	0.3444
85	0.2722	0.1688	0.3344	0.2009	0.3494	0.2235	0.3688	0.2422	0.3906	0.2626
8	0.1886	0.1174	0.2279	0.1410	0.2362	0.1537	0.2493	0.1626	0.2643	0.1742
26	0.0974	0.0559	0.1160	0.0685	0.1195	0.0726	0.1262	0.0746	0.1339	0.0788
0.	0.0000	0.0000	0.000.0	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000

* X/B = NON DIMENSIONAL AXIAL COORDINATE USING THE AXIAL CHORD
YS/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE SUCTION SURFACE USING THE AXIAL CHORD
YP/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE PRESSURE SURFACE USING THE AXIAL CHORD

TABLE III (CONTINUED) FOURTH STAGE BLADE NON-DIMENSIONAL AIRFOIL COORDINATES

90,2	RO	ROOT	QUARTER ROOT	TER	MEAN	NA	QUARTER	ARTER TIP	TIP	<u>a</u>
٥/٧	YS/B*	YP/B*	YS/B	YP/B	YS/B	YP/B	YS/B	YP/B	YS/B	YP/B
00	1945	1946	0.2467	0.2467	0	3,55	0010	2 00200	0000	
8	0.2692	0.2238	0.3470	0.2908	0.5388	0.4840	0.7525	0.7003	0.9453	0.8880
01:	0.3426	0.2780	0.4299	0.3418	0.6127	0.5236	0.8143		0.9878	0.9307
.15	0.4066	0.3208	0.4990	0.3770	0.6710	0.5494	0.8609	0.7516	1.0191	0.9440
.20	0.4611	0.3540	0.5558	0.4014	0.7161	0.5641	0.8942	0.7578	1.0393	0.9458
.25	0.5057	0.3790	0.6011	0.4176	0.7493	0.5703	0.9152	0.7545	1.0487	0.9366
8.	0.5403	0.3966	0.6356	0.4270	0.7717	0.5694	0.9248	0.7431	1.0474	0.9170
38.	0.5647	0.4074	0.6598	0.4306	0.7839	0.5624	0.9237	0.7247	1.0356	0.8879
.40	0.5789	0.4120	0.6739	0.4290	0.7863	0.5500	0.9120	0.7000	1.0133	0.8499
.45	0.5829	0.4105	0.6781	0.4224	0.7790	0.5324	0.8902	0.6693	0.9806	0.8042
.50	0.5766	0.4032	0.6724	0.4112	0.7621	0.5100	0.8581	0.6332	0.9373	0.7514
.55	0.5601	0.3902	0.6565	0.3954	0.7352	0.4829	0.8157	0.5919	0.8834	0.6925
99.	0.5336	0.3715	0.6301	0.3750	0.6979	0.4511	0.7626	0.5454	0.8189	0.6282
.65	0.4972	0.3471	0.5927	0.3499	0.6495	0.4146	0.6984	0.4939	0.7434	0.5590
02.	0.4512	0.3167	0.5436	0.3199	0.5888	0.3731	0.6224	0.4373	0.6570	0.4857
.75	0.3958	0.2802	0.4817	0.2844	0.5142	0.3263	0.5341	0.3757	0.5606	0.4087
.80	0.3313	0.2371	0.4657	0.2430	0.4252	0.2738	0.4360	0.3087	0.4564	0.3285
.85	0.2582	0.1868	0.3158	0.1944	0.3256	0.2147	0.3315	0.2363	0.3468	0.2455
œ.	0.1778	0.1287	0.2160	0.1372	0.2200	0.1479	0.2232	0.1581	0.2336	0.1600
36:	0.0916	0.0616	0.1102	0.0683	0.1114	0.0713	0.1128	0.0735	0.1181	0.0723
1.0	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.000.0

• X/B = NON DIMENSIONAL AXIAL COORDINATE USING THE AXIAL CHORD
YS/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE SUCTION SURFACE USING THE AXIAL CHORD
YP/B = NON DIMENSIONAL TANGENTIAL COORDINATE FOR THE PRESSURE SURFACE USING THE AXIAL CHORD

TABLE IV EGV AIRFOIL DATA

AVG. PCT. SPAN	0	15%	45%	75%	90%
DEFINING RADIUS \sim CM.	19.116	21.039	24.882	28.722	30.645
DEFINING RADIUS \sim IN.	7.526	8.283	9.796	11.308	12.065
α_1^{\bullet}	30.52	30.27	32.99	36.53	38.96
~ · · · · · · · · · · · · · · · · · · ·	105.87	106.95	107.18	106.16	104.42
$rac{lpha_2}{ heta}$	75.35	76.68	74.19	69.62	65.45
B CM	8.636	8.636	8.636	8.636	8.636
; IN.	3.400	3.400	3.400	3.400	3,400
τ/Β	0.4967	0.5467	0.6465	0.7464	0.7963
LER CM.	0.0208	0.0251	0.0358	0.0480	0.0549
IN.	0.0082	0.0099	0.0141	0.0189	0.0216
TER CM.	0.0239	0.0264	0.0312	0.0361	0.0383
IN.	0.0094	0.0104	0.0123	0.0142	0.0151

TABLE V

EXIT GUIDE VANE NONDIMENSIONAL COORDINATES

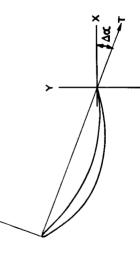
T/8 •		%0	16	% 91	4	45%	32	75%	100%	%
-	S/B _T *	P/B _T *	S/B _T	P/B _T	S/8 _T	P/B _T	S/B _T	P/B _T	S/B _T	P/B _T
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.052	0.0221	0.0542	0.0214	0.0569	0.0172	0.0586	0.0121	0.0588	0.0087	0.0576
0.110	0.0506	0.0941	0.0500	0.0981	0.0432	0.0995	0.0345	0.0984	0.0284	0.0957
0.160	0.0724	0.1221	0.0719	0.1267	0.0635	0.1280	0.0524	0.1260	0.0443	0.1222
0.211	0.0911	0.1451	0:0907	0.1503	0.0310	0.1513	0.0630	0.1485	0.0584	0.1439
0.261	0.1064	0.1638	0.1062	0.1694	0.0956	0.1702	0.0811	0.1667	0.0704	0.1613
0.311	0.1190	0.1783	0.1189	0.1842	0.1076	0.1847	0.0920	0.1807	0.0804	0.1748
0.362	0.1237	0.1890	0.1287	0.1950	0.1170	0.1954	0.1006	0.1910	0.0883	0.1847
0.419	0.1357	0.1960	0.1368	0.2022	0.1238	0.2023	0.1070	0.1975	0.0943	0.1908
0.462	0.1400	0.1994	0.1402	0.2056	0.1283	0.2055	0.114	0.2004	0.0985	0.1935
0.513	0.1417	0.1993	0.1421	0.2054	0.1304	0.2050	0.1137	0.1997	0.1009	0.1926
0.563	0.1409	0.1956	0.1414	0.2015	0.1301	0.2009	0.1139	0.1954	0.1014	0.1883
0.613	0.1375	0.1833	0.1331	0.1939	0.1274	0.1931	0.1120	0.1874	0.1001	0.1804
0.664	0.1313	0.1773	0.1320	0.1826	0.1221	0.1816	0.1078	0.1759	9960.0	0.1690
0.714	0.1220	0.1628	0.1228	0.1676	0.1139	0.1665	0.1007	0.1609	0.0905	0.1543
0.764	0.1093	0.1447	0.1103	0.1497	0.1023	0.1498	9060'0	0.1425	0.0815	0.1364
0.815	0.0932	0.1228	0.0940	0.1266	0.0872	0.1253	0.0773	0.1206	0.0695	0.1152
0.865	0.0732	0.0968	0.0739	0.0993	0.0634	0.0828	0.0605	0.0949	0.0543	0.0905
0.915	0.0483	0.0661	0.0492	0.0633	0.0453	0.0676	0.0398	0.0650	0.0355	0.0620
0.951	0.0233	0.0411	0.0234	0.0425	0.0258	0.0423	0.0223	0.0403	0.0197	0.0390
1.00	0.0	0.0	0.0	0.0	0.0	0.0	0:0	0.0	0:0	0.0
										

T/B $_{
m T}$ = COORDINATE ALONG THE TRUE CHORD, NONDIMENSIONALIZED BY THE TRUE CHORD

 $NS/B_{T}=SUCTION$ SURFACE COORDINATE NORMAL TO THE TRUE CHORD, NONDIMENSIONALIZED BY THE TRUE CHORD

 $\mathsf{NP/B_T}$ = pressure surface coordinate normal to the true chord, nondimensionalized by the true chord

NONDIMENSIONAL CORRDINATES ARE GIVEN RELATIVE TO THE AIRFOIL TRUE CHORD WHICH IS STAGGERED FROM THE AXIAL DIRECTION BY A $\Delta\alpha$ OF $_{-21.8}$, $_{-21.4}$, $_{-29.9}$, $_{-18.7}$ AND $_{-18.3}$ FOR THE 0%, 15%, 45%, 75% AND 100% SECTIONS RESPECTIVELY. NOTE:



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